

Tersuson

SERVICE MANUAL

SPECIFICATION AND DATA TRACTORS TYPE TE-A20 and TE-D20

This specification initially gives data of the basic normal-width tractor, less engine; specifications of engines then follow.

Details applicable to tractors which deviate from this standard, e.g. narrow-width models, are given in Section T, under the title of 'Special Type Tractors.'

TRACTOR, LESS ENGINE

Overall Dimensions.

Wheel Base 70" (1778 mm.)

Track: Front adjustable in 4" steps (102 mm) from 4ft. to 6ft. 8 ins. (1219 to 2032 mm.)
,, Rear ,, ,, ,, ,, 4ft. to 6ft. 4 ins. (1219 to 1930 mm.)

Turning circle: with brakes 16 ft. (4876 mm) without brakes 20 ft. (6095 mm).

Ground clearance 13" (330 mm.)

Overall length 9 ft. 7 ins. (2921 mm.)

Overall width 5 ft. 4 ins. (1625 mm.)

Overall height 4 ft. 8 ins. (1320 mm.) Weight (dry) 2500 lb. (1134 kg). Tyres front 4.00×19 . Tyres rear 10.00×28 .

Fill-Up Data.

Engine Sump 12 pints (6.8 litres) Air Cleaner 3 pint (.45 litres) Cooling System 20 pints (11.3 litres) Transmission casing 5 gallons (22.8 litres)
Belt Pulley ½ pint (.28 litres)
Fuel tank-TE-A20: 8 gallons (36 litres)
-TE-D20: 7 gallons (31.5 litres) and
I gallon (4.5 litres)

Steering Box 5 pints. (2.9 litres)
Front Hubs (each) & pint (.30 litres)
Tyre pressures: Front 26 lb. per sq. inch
Rear 12 lb. per sq. inch

Component Details Hydraulic Pump.	Dimensions New		Clearance New		Permissible Worn Clearance or Dimension		Remarks
	Ins.	mms.	ins.	mms.	Ins.	mms.	
Housing Bore for Control Valve Bush	.999 1.000	25.375 25.400	001	025			
Control Valve Bush Ext. Dia.	1.001 1.0015	25.425 25.438	—.0025	—.064			
Control Valve Bush Int. Dia.	.5925 .5030	15.959 15.062	.0005 .0012	.013 .031			
Control Valve Dia.	.5918 .5920	15.032 15.037	.0012	.031			

Torque Loadings—Valve Chamber Clamp Bolts 70 lbs. ft. (9.675 kg.m). Pump Assembly Securing Bolts 45 lbs. ft. (6.25 kg.m).

Hydraulic Lift Assy.

Fork retraction spring—Free length inside hooks 4.88" (123.952 mm). Length under 12 lbs. (5.443 kg) load 6.94" (176.276 mm).

Hydraulic Cylinder Bore	2. 4 995 2.5010	63.487 63.525	.0015 .0040	.038 .102	.006	.152	
Hydraulic Cylinder Piston Dia.	2. 4 97 2. 4 98	63.424 63.449	.0040	.102			
Piston Ring Groove Width	.1255 .1265	3.188 3.213	.0015 .0035	.038 .089	.005	.126	
Piston Ring Width	.124 .123	3.150 3.124	.0033	.007			
Piston Ring Gap (Closed)	.007 .013	.178 .330			.018	.457	
Linkage Check Chain Length.	8.015 7.980	203.581 202.692					Length of check chain taken between inside faces of end links.

Component Details		nensions New		rance ew		ole Worn r Dimension	Rem	arks
	ins.	mms.	Ins.	mms.	Ins.	mms.		
Check Chain Shackle Length	1.189 1.205	30.201 30.607					This dimension pin hole cen face.	
Check Chain Assy. Total Length	10.358 10.425	263.093 264.795			10.7	271.76	This length hole centres.	between pir
Transmission.								
Shifter Mechanism.								
Shifter Rail Dia.	.7465 .7475	18.961 18.987	.0015	.038	.006	.152		
Shifter Rail Bore in Casing	.749 .750	19.025 19.050	.0035	.089			♣	
Details. sl	nims up to	12 ± 1 lb. Loa a maximum of length .832" (2	5/16" (7.938 m	m) giving ma	ax. load of 27 lb	s. (12.247 kg)). Free length	5″ (1.588 mm) 1.571″ (39.903
Thickness of Shifter Forks at Pressure Faces.	.372 .368	9.449 9.347	.008	.203	.025	.634		
Width of Groove in Coupling Connecto	.380 rs .384	9.652 9.754	.016	.406				
Mainshaft.								
First Gear Bushing Bore.	2.0620 2.0635	52.375 52.413	.0015	.038	.007	.177		
Mainshaft Dia. at Position of 1st Gear	2.0605 - 2.0583	52.337 52.281	.0052	.132		,		
2nd Gear Bushing Bore.	2.0620 2.0635	52.375 52.413	.0015	.038	.007	.177	**	
External Dia. of Bearing Connector	2.0605 2.0683	52.337 52.281	.0052	.132				
Countershaft.								
3rd. Gear Bushing Bore.	2.0620 2.0635	52.375 52.413	.0015	.038	.008	.203		
Ext. Dia. of Counter- shaft 3rd. Gear Bus	2.0605 h 2.0583	52.337 52.281	.0052	.132				
4th. Gear Bushing Bore	2.0635 2.0620	52.413 52.375	.0052	.132	.007	.177		
Ext Dia. of Connector Bearing.	2.0605 2.0583	52.377 52.281	.0015	.038				
Reverse Gear Bushing Bore	1.1250 1.1256	28.575 28.590	.002 .003	.051 .076	.008	.203	ų ,	
Reverse Shaft Dia.	1.123 1.122	28.52 4 28.499	.003	.070				
End Float—Main & Countershaft.			See f	Remarks		:	Fit shims behing bearing retaine Shaft bearing give preload of ins. (.081—.138 main and cour	r and P.T.O. support to 7 to 12 lbs. 3 kg.m). on

Component Details	Dimensions New		Clearance New		Permissible V Clearance or Di		Remarks
	Ins.	mms.	Ins.	mms.	Ins:	mms.	
Backlash(A)—Sliding Mating Teeth on Gear	Coupling ar Wheels	d	(B)—Sliding cou	pling and connectors-	(C)— — Gear te	eth	
008″ .203 mm 010″ .254 mm			.0005″ .0015″	.013 mm .038 mm	004′		

Power Take-off Shaft.

Rear splines 6 × 1.92" (48.77 mm) long × 1.121" (28.47 mm) dia. × .922" (23.42 mm) dia. × .275" (6.99 mm) wide. 1.123" (28.52 mm) .932" (23.67 mm) .277" (7.04 mm) $21/64^{\prime\prime}$ (8.33 mm) dia. hole at distance of $\frac{1}{2}^{\prime\prime}$ (12.7 mm) from shaft end. End Cover Int. Dia. $2\frac{1}{2}^{\prime\prime}$ (63.5 mm).

Clutch.

Release Shaft Dia.	.997 .996	25.32 4 25.298	.004 .0065	.102 .165	.010	.254	
Bush Bore.	1.001 1.0025	25.425 25.464	.0003	.105			
Clutch Springs.	9 green spri	ngs each of 105	lb. (47.627 kg) to 115 lb. (5	(2.199 kg).		Orange spri

9 orange springs each of 90 lb. (40.823 kg) to 100 lb. (45.358 kg).

Orange springs superseded green springs after Tractor Serial No. 32872.

Free movement of pedal should be $\frac{3}{8}$ " (9.5 mm). This dimension taken between upper side of pedal and underside of footrest bracket. Movement of release lever ends $\frac{1}{2}$ " (13 mm). Variation in release lever height should not exceed .015" (.381 mm).

Rear Axle.

Backlash — Crown wheel and pinion .004" (.102 mm). Half shaft end float .008" (.203 mm). .025" (.634 mm). .020" (.508 mm). .025" (.634 mm). .025" (.033 mm). .020" (.033 mm). .020" (.508 mm).

Front Axle.

Centre Trunnion Bushing Int. Dia. Fitted.	1.756 1.764	44.602 44.806	.008 .017	.203 .432	.035	.088
Centre Pin Dia.	1.747 1.748	44.347 44.399	.017	.732		
Bore of Outer Axle for Spindle Bushes	1.3735 1.3745	34.887 34.912	±.001	±.025		
Ext. Dia. Spindle Bushes.	1.3735 1.3745	34.887 34.912				
Int. Dia. Spindle Bushes	1.249 1.250	31.725 31.750	.003 .005	.076 .127	.010	.254
Spindle Dia.	1.246 1.245	31.648 31.623	.003	.127		

Steering.

Backlash—Screw adjustment against rear faces of segments to give minimum backlash without binding.

Distances between ball centres and vertical plane through drop arm crankshaft centre 2.17" (55.12 mm) with steering wheel in straight ahead position.

Pulley Attachment.

Pulley width $6\frac{1}{2}$ " (165 mm) Dia. 9" (229 mm). Gear Ratio to P.T.O. Shaft 1.86 to 1. Backlash between driving gears .004" (.102 mm) .020" (.508 mm)

PETROL ENGINE, PART No. 57963

(Manufactured by the Standard Motor Co.)

ENGINE—80 mm bore, fitted to tractors Type TE-A20, TE-C20.

Stroke 92 mm. Piston Displacement 112.9 cu. ins. (1850 c.c.)

Compression ratio 5.77 to 1.

Maximum belt horse power—23.9.

Tightening Torque—Cylinder Head Nuts 60 to 65 lbs. ft. (8.25—8.95 kg. metres).

Big End Nuts 42 to 46 lbs. ft. (5.8—6.4 kg. metres).

Main Bearing Nuts 90 to 100 lbs. ft. (12.4—13.8 kg. metres). Flywheel Cap Screws 42 to 46 lbs. ft. (5.8—6.4 kg. metres).

Component Details	Dimensions New		Cleara Nev		Permissible Worn Clearance or Dimension		Remarks
	Ins.	mms.	Ins.	mms.	¹ Ins.	mms.	
Crankshaft. Journal Diameter	2.4795 2.4790	62.979 62.967	.0025	.064	.006	.152	Similar tolerances for reground crankshaft to .020",
Bearing Diameter (Fitted)	2.4815 2.4805	63.030 63.015	.0010	.025	dry		.030", .040" (.508, .762, 1.016 mm) undersize.
Crankshaft End Float.	•						
Centre Journal Length.	1.7507 1.7 4 98	44.468 44.445	.0117	.297	.010	.254	Crankshaft end float controlled by thickness of
Centre Bearing Cap width $+$ 2 thrust washers.	1.7450 1.7390	44.323 44.171	.0048	.122	dry		thrust washers.
Big End. Crankpin Diameter	2.0861 2.0866	52.987 53.000					Similar tolerances for re-
Bearing Diameter	2.0985 2.0872	53.302 52.015	.0024 .0006	.061 .015	.006	.152	ground crankshaft to .020", .030", .040" (.408, .762, 1.016 mm) undersize.
		32.013			9 1		
Connecting Rod End Float Crankpin Length	i. 1.1890 1.1870	30.201 30.150	0115	202			
Con-Rod Width	1.1795 1.1775	29.959 29.909	.0115 .0075	.292 .191			
Ovality—Journals & Crankpins.					0.002	.051	Minimum diameter to be such that the permissible
Taper—Journals & Crankpins.					0.002	.051	worn clearance for bearings is not exceeded.
Small End. Bore for Bush	1.0000 .9995	25. 4 25.387	—.003 5	+ .09			Heat piston in boiling
Bush, Ext. Dia.	1.0045 1.0035	25.514 25.489	0050	—.13			water for removal and fitting of gudgeon pin.
Bush, Int. Dia.	.8752 .8738	22.230 22.220				₩	
Gudgeon Pin, Dia.	.87510 .87485	22.228 22.221	÷.00035 —.00030	+.009 008			These clearance figures taken at 68°F.
Gudgeon Pin Holes in Piston	.8853 .87505	22.233 22.226	+ .00045 00005	+.011 —.001			

Component Details	Dir	mensions New		earance New	Permissib Clearance or		n Remarks
Pistons & Sleeves.	Ins.	mms.	Ins.	mms.	Ins.	mms.	
Piston Dia.—(Thrust Side Top Skirt)	3.1461 3.1472		.0028 .0034	.071 .086			Sleeves and pistons graded F.G.H. in steps of .0004' (.010 mm).
Sleeve Bore (Parallel)	3.1492 3.1503	79.99 80.018	.0013	.033			
Piston Dia. (Thrust Side Bottom Skirt)	3.1476 3.1487	79.949 79.977	.0019	.048			
Top Land Diameter	3.133 3.131	79.578 79.527	.0162 .0193	.412 .490			Piston fitted with three rings above gudgeon pin.
Ring Groove Width Top and 2nd.	.0957 .0947	2.431 2.405	.0030	.076	.005	.127	On engines S101E to S56962E a plain bottom
Compression Ring Width Top & 2nd.	.0937 .0927	2.380 2.355	.0010	.025	.003	.127	scraper ring fitted below gudgeon pin.
Ring Groove Width (3rd.)	.1895 .1885	4.813 4.788	.0030	.076	005	127	Similar tolerances for over- size pistons +.020" (.508
Scraper Ring Width (3rd.)	.1875 .1865	4.763 4.737	.0010	.025	.005	.127	mm). Oversize rings + .010" (.245 mm) + .020" (.508 mm) + .030" (.762 mm).
Ring Groove Width (4th.)	.1580 .1570	4.013 3.987	0030	074			Replacement sleeves available as standard size, and rebored + .020" (.508 mm).
Scraper Ring Width (4th.)	.1560 .1550	3.962 3.937	.0030 .0010	.076 .025	.005	.127	
Ring Gap (Closed)			.010 .006	.25 .15			
Clearance Between:					•		
Sleeve & Upper Block			.045 .015	1.1 43 .381			Dimensions taken respectively at top flange and
Sleeve & Lower Block			.003 .0005	.076 .013			spigot of sleeve.
Stand-out of Sleeve			.003 .0055	.076 .1 4 0			Desired clearance when assembled.
Water Pump & Thermost	at.						
Housing Bore for Bearing	1.1813 1.1807	30.005 29.990	+.0007	+ .018			
Bearing Case, Ext. Dia.	1.1811 1.1806	30.000 29.987	—.000 4	—.010			
Oil Pump. Approximate capacity at	: 50 lbs. pe	r square inch	(3.52 kg/sq. cn	n.) is 3.95 gallo	ons (16.94 litres)) per minut	e at 2,000 r.p.m. (Engine)
Outer Rotor, outside dia.	1.59 8 1.599	40.589 40.615	.001	.025			
Housing, internal dia.	1.601 1.600	40.665 40.640	.003	.075			
Rotor depth—outer and inner:	0.9995 0.9985	25.387 25.362	.0005	.013			A combined worn clearance
Housing depth	1.001 1.000	25.403 25.400	.0015	.038		•	of .004" (.101mm) indicates need of cover and housing face lapping.

Component Details		nensions New		rance ew	Permissib Clearance o		Remarks
Inner rotor, major dia.	Ins. 1.171 1.172	mms. 29.743 29.769	Ins.	mms.	Ins.	mms.	
Inner rotor, minor dia.	.729 .731	18.517 18.567					
Clearance on rotors			.004 .0005	.102 .013			Where clearance exceeds .010" (.253 mm) new parts should be fitted.
Camshaft.							
Front Journal Dia.	2.0590 2.0595	52.299 52.311	.0045	.114	.0065	.164	Max. wear on camshaft
Bore in Block	2.0635 2.0620	52.413 52.375	.0025	.051			journals .003" (.076 mm) and .0035 (.088 mm) in cylinder block.
2nd Journal Dia. 3rd. ,, ,, Rear ,, ,,	1.71575 1.71525	43.580 43.567	.0045 .0025	.114 .051	.0065	.164	
Bore in Block	1.71975 1.71825	43.683 43.645	.0023	.031			
Locating Groove	.1885 .1865	4.788 4.737	.0065	.165			This clearance determines
Locating Plate	.1835 .1820	4.661 4.623	.003	.076			camshaft end float.
Tappets & Valves. Tappet Bore in Block	.9380 .9373	23.825 23.807	.0013	.033			
Tappet Dia.	.9371 .9367	23.802 23.792	.0002	.005			
Valve Tip Clearance Inlet Exhaust			.010 .012	.254 .305			
Valve Guide Bore Dia.	.313 .312	7.950 7.925					
Inlet Valve Stem Dia.	.311 .310	7.899 7.87 4	.001 .003	.025 .076			
Exhaust Valve Stem Dia.	.309 .308	7.849 7.823	.003 .005	.076 .127			
Guide projection above spring seat.	9/16″	14.3					

Valve seating angle on valve head 45°. Valve seat angle in cylinder head 44½°.

Valve Springs. Free length 1.716" (43.586 mm). Fitted load 38 lbs \pm 2 lb. (17.237 kg \pm .907 kg). Fitted length 1.25 (31.75 mm). Full lift load 60 lb. (27 kg) approx.

Flywheel.

Spigot dia. (for Starter Gear Ring)	13.406 13.403	340.512 340.436		
3,			031	787
			023	—.58 4
Starter Gear Ring	13.380	339.852		
Inside Dia	13.375	339.725		

Flywheels balanced individrlywheels balanced individ-ually. Held to crankshaft by 4 set screws locked in pairs. Single dowel. Locating holes in flywheel 90° apart, in crankshaft 180° apart.

Run-out of clutch contact face at outer dia. should not exceed .003" (.076 mm),

Clearance between starter pinion and ring gear, Engine Serial No.SIE—S67028E, .156" (3.962 mm). Engine Serial No. S67029E onwards .114" (2.896 mm). Face-up starter mounting flange or fit shims to suit.

Component Details		Dimensions New			Clearance New		Permissible Worn Clearance or Dimension		Remarks
		Ins.	mms.	Ins.	mms.	Ins.	mm	ıs.	
Carburettor. Zenith Type 24 T	<u>2.</u>	Zenith T	Гуре 24Т—	-2 (Min./Max. Adj.	Jet)	н	olley.		
Choke Tube	17	Chol	ke Tube	17	Discha	arge Nozzle.		.104″ .040″	(2.642 mm) with 4 hole (1.016 mm) dia.
Main let	120	Main	ı jet	100	Float	Needle Seat		.081"	(2.057 mm), dia.
Adj. Needle	12	Adj.	Needle	1.00 drilled				.083″	(2.108 mm) dia.
S.R. Jet	50	S.R.	Jet	50	Main J	l et		.035″	(.889 mm) dia.
Progression	120	Prog	ression	120	High :	Speed Bleed		.0293	" (.744 mm) dia.
Needle Seating	1.5 mm	Nee	dle Seating	1.5 mm	Upper	r'Idle Restrictio	on	.046"	(l.168 mm) dia.
Air Jet	2.0	Petro	ol level at	4′ 6″	Idle D	ischarge Hole		.052″	(1.321 mm) dia.
,			Head	15 mm	Secon	d Idle Discharg	e Hole	.046"	(1.168 mm) dia.
S.R. Bottom Feed	1.5	Inter	r-con	Imm drilled	Ventu	ıri		21/32	" (16.669 mm) dia.
• • • • • • • • • • • • • • • • • • • •		Air	let	2.0	Fuel l	_evel at 3/4 (.34	H kgm)		1.01 ± 1.01 1.01 1.01 1.01 1.01
				s Stamped M-M	fuel	pressure	• ,		n) to top face of fuel bow
				•	Float	cut-off position		7/16″ bet	(11.13 mm) measure ween upper casting fac douter float top.

Governor.

Governor lever spring: Free length: inside hooks 3.8" (96.5 mm). End Play .005" (.127 mm) Rate: 18 lbs/in + 5%. No of coils: 26. .010" (.254 mm)

Load at 1" (25.4 mm) deflection: 25 lbs. (11.34 kg) + 1 lb. (.454 kg.) Initial wound-in load: 7 lbs. (3.175 kg).

Control Rod: Free length: inside hooks 2.687" (68.25 mm). Rate: 64 lbs./in = 5%. No of coils: 11½.

Compensating Spring: Load at $\frac{1}{2}$ " (12.7 mm) deflection: 38 lbs. (17.237 kg) $\pm 1\frac{1}{2}$ lbs. (.681 kg). Initial wound-in load: 6 lbs. (2.722 kg).

VAPORISING OIL ENGINE, PART No. 500038

(Manufactured by the Standard Motor Co.)

ENGINE—85 mm bore, fitted to tractors Type TE-D20, TE-E20.

Stroke 92 mm. Piston Displacement 127 cu. ins. (2088 c.c.)

Compression ratio 4.8 to 1.

Maximum belt horse power-23.9.

Tightening Torque—Cylinder Head Nuts 60 to 65 lbs. ft.
(8.25—8.95 kg. metres).
Big End Nuts 42 to 46 lbs. ft.
(5.8—6.4 kg. metres).

Main Bearing Nuts 90 to 100 lbs. ft. (12.4—13.8 kg. metres). Flywheel Cap Screws 42 to 46 lbs. ft. (5.8—6.4 kg. metres).

Component Details		ensions Iew	Cleara Nev		Permissib Clearance or		Remarks
Crankshaft.	Ins.	mms.	Ins.	mms.	Ins.	mms.	
Journal Diameter	2.4795 2.4790	62.979 62.967	.0025 .0010	.064 .025	.006 dry	.152	Similar tolerances for reground crankshaft to .020",
Bearing Diameter (Fitted)	2.4815 2.4805	63.030 63.015		.025	dry		.030", .040" (.508, .762, 1.016 mm) undersize.
Crankshaft End Float. Centre Journal Length.	1.7507 1.7498	44.468 44.445	.0117	.297	.010	.254	Crankshaft end float con- trolled by thickness of
Centre Bearing Cap width $+$ 2 thrust washers.	1.7450 1.7390	44.323 44.171	.0048	.122	dry		thrust washers.
Big End. Crankpin Diameter	2.0861 2.0866	52.987 53.000	.0024	.061	.006	.152	Similar tolerances for reground crankshaft to .020",
Bearing Diameter	2.0985 2.0872	53.302 52.015	.0006	.015			.030", .040" (.508, .762, 1.016 mm) undersize.
Connecting Rod End Float. Crankpin Length	1.1890 1.1870	30.201 30.150	.0115	.292			
Con-Rod Width	1.1795 1.1775	29.959 29.909	.0075	.191			
Ovality—Journals & Crankpins.					0.002	.051	Minimum diameter to be such that the permissible
Taper—Journals & Crankpins.					0.002	.051	worn clearance for bearings is not exceeded.
Small End. Bore for Bush	1.0000 .9995	25.4 25.387	 .0035	+.09			Heat piston in boiling
Bush, Ext. Dia.	1.0045 1.0035	25.514 25.489	+.0050	13			Heat piston in boiling water for removal and fitting of gudgeon pin.
Bush, Int. Dia.	.8752 .8738	22.230 22.220	00035	. 000			TI
Gudgeon Pin, Dia.	.87510 .87485	22.228 22.221	+.00035 00030 +.00045	+.009 008 +.011			These clearance figures taken at 68°F.
Gudgeon Pin Holes in Piston	.8853 .87505	22.233 22.226	00005	<u></u> .001	d		

Component Details		nensions New		arance lew		ble Worn r Dimension	Remarks
Distance 9 Classes	Ins.	mms.	Ins.	mms.	ins.	mms.	
Pistons & Sleeves. Piston Dia.—(Thrust Side Top Skirt)	3.3429 3.3438	84.905 84.933					Sleeves and pistons graded F.G.H. in steps of .0004" (.010 mm).
Sleeve Bore (Parallel)	3.3460 3.3471	84.988 85.016					
Piston Dia. (Thrust Side Bottom Skirt)	3.3442 3.3453	84.943 84.971					
Top Land Clearance			.017 .019	.432 .483			
Ring Groove Width Top 2nd and 3rd.	.0797 .0807	2.024 2.050	.0030	.076 .025	.005	.127	
Compression Ring Width Top 2nd & 3rd	.0787 0777	1.999 1.974	.0010	.023			
Ring Groove Width (4th.)	.1895 .1885	4.813 4.788	.0030	.076	.005	.127	Similar tolerances for oversize pistons $+.020$ " (.508 mm).
Scraper Ring Width (4th.)	.1875 .1865	4.763 4.737	.0010	.025			Oversize rings + .010" (.245 mm) + .020" (.508 mm) + .030" (.762 mm).
Ring Groove Width (5th.)	.1580 · .1570	4.013 3.987	.0030	.076	.005	.127	Replacement sleeves available as standard size, and rebored + .020" (.508 mm).
Slotted Scraper RingWidt (5th.)	h .1560 .1550	3.962 3.937	.0010	.025			
Ring Gap (Closed)			.010 .006	.25 .15			
Clearance Between :							
Sleeve & Upper Block			.045 .015	1.1 43 .381			Dimensions taken respectively at top flange and spigot of sleeve.
Sleeve & Lower Block			.003 .0005	.076 .013			spigot of siecve.
Stand-out of Sleeve			.003 .0055	.076 .140			Desired clearance when assembled.
Water Pump & Thermosta	t.						
Housing Bore for Bearing	1.1813 1.1807	30.005 29.990	+.0007	+ .018			
Bearing Case, Ext. Dia.	1.1811 1.1806	30.000 29.987	—.000 4	—.010			
Oil Pump. Approximate capacity at	50 lbs. pe	er square incl	h (3.52 kg/sq. cr	n.) is 3.95 gal	lons (16.94 litr	es) per minut	te at 2.000 r.p.m. (Engine)
Outer Rotor, outside dia.	1.598 1.599	4 0.589 4 0.615	.001	.025			
Housing, internal dia.	1.601 1.600	40.665 40.6 4 0	.003	.075			
Rotor depth—outer and inner:	0.9995 0.9985	25.387 25.362	.0005	.013			A combined worn clearance
Housing depth	1.001 1.000	25.403 25.400	.0015	.038			of .004" (.101mm) indicates need of cover and housing face lapping.

Component Details		nensions New		rance ew	Permissibl Clearance or		Remarks
Inner rotor, major dia.	Ins. 1.171 1.172	mms. 29.743 29.769	Ins.	mms.	Ins.	mms.	
Inner rotor, minor dia.	.729 .731	18.517 18.567					
Clearance on rotors	./31	10.307	.004 .0005	.102 .013			Where clearance exceed .010" (.253 mm) new part should be fitted.
amshaft.							should be litted.
Front Journal Dia.	2.0590 2.0595	52.299 52.311	.0045	.114	.0065	.164	Max. wear on camshaf
Bore in Block	2.0635 2.0620	52.413 52.375	.0025	.051			journals .003" (.076 mm and .0035 (.088 mm) is cylinder block.
2nd Journal Dia. 3rd. " " Rear " "	1.71575 1.71525	43.580 43.567	.0045	.114	.0065	.164	
Bore in Block	1.71975 1.71825	43.683 43.645	.0025	.051			
Locating Groove	.1885 .1865	4.788 4.737	00/5	145			TI:
Locating Plate	.1835	4.661 4.623	.0065 .003	.165 .076			This clearance determine camshaft end float.

ppets & Valves. Tappet Bore in Block	.9380 .9373	23.825 23.807	.0013	.033			
			.0002	.005			
Tappet Dia.	.937 I .9367	23.802 23.792					
Valve Tip Clearance Inlet Exhaust			.010 .012	.254 .305			
Valve Guide Bore Dia.	.313 .312	7.950 7.925					
Inlet Valve Stem Dia.	.311 .310	7.899 7.874	.001 .003	.025 .076			
Exhaust Valve Stem Dia.	.309 .308	7.849 7.823	.003 .005	.076 .127			
Valve Head Diameter : Inlet	1.176 1.172	29.570 29.769					
Exhaust	1.051 1.0 4 7	26.695 26.594					
Guide projection above spring seat.	9/16*	14.3					

Valve seating angle on valve head 45°. Valve seat angle in cylinder head 44½°.

Free length 1.716" (43.586 mm). Fitted load 38 lbs \pm 2 lb. (17.237 kg \pm .907 kg). Fitted length 1.25 (31.75 mm). Full lift load 60 lb. (27 kg) approx.

Flywheel.

Spigot dia. (for 13.406 Starter Gear Ring) 13.403 340.512 340.436 --.031 --.023 -.787 **—.584** 13.380 339.852 Starter Gear Ring 13.375 339.725 Inside Dia.

Flywheels balanced individ-Hywheels balanced individually. Held to crankshaft by 4 set screws locked in pairs. Single dowel. Locating holes in flywheel 90° apart, in crankshaft 180° apart.

Run-out of clutch contact face at outer dia. should not exceed .003" (.076 mm), Clearance between starter pinion and ring gear, Engine Serial No.SIE—S67028E, .156" (3.962 mm). Engine Serial No. S67029E onwards .114" (2.896 mm). Face-up starter mounting flange or fit shims to suit.

Carburettor.

Zenith Type 24T—2 (Min./Max. Adj. Jet) Choke Tube: 17. Main Jet: 105. Adj. Needle: 1.25 drilled. S.R. Jet: 60. Progression: 120. Needle Seating: 1.5 mm. Petrol level at 4' 6" Head: 15 mm. Inter-con: Imm drilled. Air Jet: 2.0. Petrol Inlet Boss and adjusting needle head Stamped V.O.

Governor.

Governor lever spring: Free length: inside hooks 3.8" (96.5 mm). End Play .005" (.127 mm) Rate: 18 lbs/in \pm 5%. No of coils: 26. .010" (.254 mm)

Load at 1" (25.4 mm) deflection: 25 lbs. (11.34 kg) \pm 1 lb. (.454 kg.) Initial wound-in load: 7 lbs. (3.175 kg). Control Rod: Free length: inside hooks 2.687" (68.25 mm). Rate: 64 lbs./in \pm 5%. No of coils: $11\frac{1}{2}$.

Compensating Spring: Load at $\frac{1}{2}$ " (12.7 mm) deflection: 38 lbs. (17.237 kg) $\pm 1\frac{1}{2}$ lbs. (.681 kg). Initial wound-in load: 6 lbs. (2.722 kg).

4.50

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DIESEL ENGINE

(Manufactured by the Standard Motor Co.)

ENGINE-3-3" (80.96 mm) bore × 4" (101.6 mm) stroke, 4 cylinders, fitted to Tractors Type TE-F20.

Displacement 127.68 cu. ins. (2092 cc.)

Compression Ratio 17:1

Firing Order 1, 3, 4, 2.

Maximum Belt Horse Power-26 at 2,000 r.p.m.

Tightening Torques: -- Cylinder Head Nuts 75 to 80 lb. ft.

Cylinder Head Nuts 75 to 80 lb. ft.

(10.4—11.1 kg.m)

Big End Nuts 65 to 70 lb. ft.

(9—9.7 kg.m)

Oil Pump Attachment 16 to 18 lb. ft.

(2.2—2.5 kg.m)

Flywheel Set Screws 90 to 100 lb. ft.

(12.4—13.8 kg.m)

Main Bearing Socket Screws 25 to 30 lb. ft.

(3.5—4.1 kg.m)

Centre Bearing Housing to Block 39 to 42 lb. ft.

(5.4—5.8 kg.m)

Clutch Fixing Screws 26 to 28 lb. ft.

(3.6—3.9 kg.m)

Injector Attachment 12 to 14 lb. ft.

(1.6—1.9 kg.m)

Component Details	Dime No	nsions ew	Cleara Nev		Remarks
Details	ins.	mms.	ins.	mms.	
Main Bearing Housings:					For checking external dia. of al Housings—break housing and
Housing Spigot Ext. Dia.	5.0615 5.0605	128.562 128.537	.0030 .0005	.076 .013	assemble on a mandrell 2.9180" 2.9183" (74.117/74.125 mm.) dia without bearings. Tighter Socket Screws 29—31 lb. ft
Bore in Cylinder Block	5.0635 5.0620	128.613 128.575	.0003	.013	(4—4.3 kg.m).
Centre					
Housing Ext. Dia.	6.8115 6.8105	173.012 172.987	0035	000	
			. 0035 . 0005	.089 .013	
Bore in Cylinder Block	6.8140 6.8120	173.076 173.025	.0003	.0.5	
Rear					
Housing Spigot Ext. Dia.	6.8735 - 6.8725	174.587 174.562	.004	. 102	
Bore in Cylinder Block	6.8765 6.8745	174.663 174.613	.001	.025	
Main Bearings.					For checking bore dia. assemble
Housing Bores, Front, Centre and Rear.	2.9165 2.9170	74.079 74.092			both halves with ring dowels fitted and tighten screws to 29—31 lb. ft. (4—4.3 kg.m).
Radial thickness of Bearings, Front, Centre and Rear	.08250 .08225	2.096 2.089			Front and rear Main Bearing Liners are identical but centre is .100" (2.54 mm.) wider.
Bearing Bore Dia. Front, Centre and Rear	2.7540 2.7530	69.952 69.926			With Bearings fitted into Housings tighten to specified torque setting.
			.0040 .0025	.102 .064	Desired clearance when assembled.
Crankshaft. ournal Dia.	2.7505	69.863			Similar tolerances for reground Crankshaft to .010", .020", .030",
•	2.7500	69.850			.040" (.254, .508, .762, 1.016 mm.) undersize.
Crankshaft End Float					
Rear Journal Length ·	I . 7507 I . 7498	44 . 468 44 . 445	.0117	. 297	
Rear Bearing Housing width	1.559 1.557	39.599 39.548	.00 48	.122	
Thrust Washer thickness	.093 .091	2.362 2.311			

Component	Dimen Ne		Clear: Ne		D. a. maraylar
Details	ins.	mms.	· ins.	mms.	Remarks
Big End.					
Crankpin Dia.	2.3115 2.3110	58.712 58.699	.0035 .0020	. 089 . 051	Similar tolerances for reground crankshaft to .010", .020", .030" 040", .060" (.254, .508, .762 1.016, 1.524 mm.) undersize
Bearing Bore Dia.	2.3145 2.3135	58.789 58.763			
Con. Rod Bore Dia.	2.4575 2.4570	62.421 62.408			For checking bearing bores— assemble Con. Rod and tighter to specified setting.
Bearing Shell thickness	.07175 .07150	1.822 1.816			to specified secting.
Connecting Rod End Float.					
Crankpin Length Con. Rod Width	I . 4390 I . 4370 I . 4305	36.551 36.500	.0105 .0065	. 267 . 165	For service purposes:— Max. permissible variation in Con. Rod total weights 1½ ozs. (42.52 gms.). Metal may be removed from web on bearing cap for fine weight adjustment. Con. Rod assembly weight
Coll. Rod Wideli	1.4285	36.284			graded — N, P, Q. S, T, U—in $I_{\frac{1}{2}}$ oz. stages.
imall End.					
Bore for Bush	1.126	28.600 28.575	0050	— .127	
Bush External Dia.	I.1300 I.1285	28.702 28.664	— .0025	064	
Bush Internal Dia.	1.0002 .9998	25.405 25.395	+.00035 00035	+ .009 — .009	Specified clearance using draw- ing sizes, but bore of Bush machined to suit Gudgeon Pin for the required fit.
Gudgeon Pin Dia.	1.00015 .99985	25.404 25.396	+.0003	+.008	Specified clearance using draw- ing sizes but desired fit of
Gudgeon Pin Holes in Piston.	1.00015 .99985	25.404 25.396	—.0003	—.008	gudgeon pin in Piston obtained by selective assembly. Heat piston in hot oil for fitting.
Pistons, Sleeves and Inserts.					
Wellworthy Type Pistons.					
Original Piston Skirt Dia. (Round and Parallel).	3.183 3.182	80.848 80.823	.0070	. 178 . 127	Ungraded up to Engine No. SA.7739E.
Sleeve Bore (Parallel).	3.1890 3.1880	81.001 80.975	.0050	.127	Replacement sleeves available as standard size only, (i.e. no provision made for reboring and fitting oversizes).
Ist Modification—Engine No. SA.7740E — SA.9205E.					
Piston Skirt Dia. — F. Grade (Round and Parallel).	3.1834 3.1829	80.858 80.846			Pistons and Sleeves graded F & G.
—G. Grade	3.1838 3.1834	80.868 80.858	.0056	. 142	
Sleeve Bore —F. Grade (Parallel).	3.1885 3.1880	80.988 80.975	.0047	. 120	Replacement Pistons and Sleeves available at standard size only (i.e. no provision for oversizes).
	3.1890	81.001			(F
G. Grade	3.1885	80.988			

Component		Dimen		Clearar New		Remarks
Details		Ne ins.	w mms.	ins.	mms.	veillat k2
ellworthy Type Pistons—co 2nd Modification—Engine N SA.9206E—SA.23082E.						
Piston Skirt Dia.—Top — (Oval ground tapered skirt		3.1833 3.1829	80.856 80.846			Alternative to B.H.B. ty Pistons.
_	G. Grade	3.1838 3.1834	80.868 80.858			Grade to be measured at top skirt thrust side.
3rd Modification—Engine N SA.23083E and future.	10.	3.1031	30.555			
Piston Skirt Dia.—Top — (Oval ground tapered skirt		3.1837 3.1833	80.866 80.856			
	G. Grade	3.1842 3.1838	80.879 80.868			
				.0052 .0043	. 132 . 110	
	F. Grade G. Grade	See 1st Modifi	cation.			
Ovality—Top of Skirt		.005 .004	. 127 . 102			
—Bottom of Skirt		.001 .000	.025 .000			
Piston Head Dia.		3.158	80.213			Piston Head Dia. up
(Parallel).	•	3.155	80.137	.02300	. 584 . 489	SA.15104E — 3.1625*/3.15 (80.328/80.252 mm.)
Cylinder Insert Lower Internal Dia.		3.17800 3.17725	80.721 80.702	.01725	.407	
H.B. Type Pistons (Introdu	ced as an a			735E).		
Original Piston Skirt Dia.—Top (Oval ground tapered skirt)		3.1838 3.1834	80.868 80.858			Grade to be measured at top Skirt Thrust side.
	G. Grade	3.1843	80.882		·	
		3.1839	80.871	.0051 .00 4 2	. 129 . 107	
	F. Grade G. Grade	See under We	llworthy type P	iston—Ist Modifi	caton.	
1st Modification—Engine N SA.21914E—SA.29605E.	o.					
Piston Skirt Dia.—Top — (Oval ground tapered Skirt		3.1842 3.1838	80.879 80.868			
	G. Grade	3.1847 3.1843	80.892 80.882			
		3.1013	00.002	. 0047 . 0038	. 120 . 096	
	F. Grade G. Grade	As above.				
2nd Modification—Engine N	10.			.0052 .0043	. 135 . 110	
SA.29606E and future. Piston Skirt Dia.—Top — (Oval ground tapered skirt)	F. Grade	3.1837 3.1833	80.866 80.856	.0043	.110	
	G. Grade	3.1842 3.1838	80.879 80.868			
Ovality—Top of Skirt		.0122	.310 .249			
—Bottom of Skirt		.0024	.061			
Disease Head Dis		.0008	.020 80.221			
Piston Head Dia. (Parallel)		3.1543	80.120	. 0237 . 01895	. 602	

Component	Dimen: Nev		Cleara Nev		Remarks	
Details	ins.	mms.	ins.	mms.	itemat ks	
Ring Groove Width (Top, 2nd & 3rd).	.0832 .0822	2.113 2.088	.0055 .0035	. 140 . 089	Piston Rings:— Three Compression Rings and one slotted Scraper Ring all above Gudgeon Pin. Chromium plated top Compression Ring—	
Compression Ring Width (Top, 2nd & 3rd)	.0777	1.974 4.069			deleted on the Wellworth Piston at Engine No. SA8309 and re-introduced at Engine N	
Ring Groove Width (4th).	. 1592	4.044	.005	. 127	SA.28867E. 2nd and 3rd Compression	
Slotted Scraper Ring Width (4th).	. 1562 . 1552	3.967 3.942	.003	.076	Rings: tapered periphery rings introduced at Engine No. SA. 28867E, marked "T" denoting the taper and must be fitted	
All rings—fitted gap			.014 .009	.356 .229	with "T" upwards.	
Lower Block Dia. for Sleeve.	3.4073 3.4068	86.546 86.532	.0021	.053 .020	For service purposes, oversize Piston Rings, +.010" (.254 mm.) only, available for fitting in	
Sleeve External Dia.	3.4060 3.4052	86.512 86.492			existing worn Cylinder Sleeve bores.	
Upper Block Recess Dia. for Inserts and Sleeve Flange.	3.65725 3.65625	92.894 92.868	.00245 .00045	.062		
Cylinder Insert Lower External Dia.	3.6558 3.6548	92.857 92.832				
Cylinder Insert Upper— Width of Slot. (See Remarks).	1 . 265 1 . 255	32.131 31.877			When upper insert is fitted in bore 3.6565" (92.875 mm.) dia. When in free state, gap in-	
Cylinder Insert Upper— Radial Thickness.	. 2335 . 2325	5.931 5.906			creases by .03" (.762 mm.) nominally.	
Gasket Cylinder Sleeve—Thickness.	.012	. 305				
Sleeve Flange Thickness (Up to Engine No. SA.23082E).	. 12575 . 12475	3.194 3.169			Slot in bottom of Sleeve.	
Sleeve Flange Thickness (Engine No. SA.23083E and future).	.21575	5.480 5.455			Without slot in bottom of Sleeve.	
Depth of recess for Cylinder Inserts in Cylinder Block.	. 9072 . 9057	23.043 23.005				
Depth of Cylinder Inserts Lower Insert	.2390 .2380	6.071 6.045			Specified Insert depths are over flats. As the mating surfaces are cones, these dimensions cannot be used directly for calculating	
Upper Insert (Up to Engine No. SA.23082E).	. 5510 . 5500	13.995 13.970			the 'nip.' Use original Upper Insert only with Sleeve Incorporating Slot;	
Upper Insert (Engine No. SA.23083E and future).	. 461 . 460	11.709 11.684			later type only with slotless Sleeve, which has the thicker flange.	
Stand out of upper insert above block (with new Sleeve gasket).			.0045 .0010	.115 .025		
Stand out of Piston at T.D.C. above top face of Cylinder Block (not insert)			+.010 003	+ . 254 — . 076		
Water Pump Housing Bore for Bearing	I . 5749 I . 57 44	40.003 39.990	+ .00061 — .00040	+.015 —.010		
Bearing Case External Dia.	I . 57480 I . 57429	40.000 39.987	.000 10	.0.0		
Spindle Dia.	. 6264 . 6256	15.905 15.893	—.0022 —.0012	056 030	Drive to impellor imparted through interference fit of impellor on spindle. Incor- porated at Engine No. SA.	
Impellor Bore	. 6245 . 6240	15.863 15.850			14655E.	

Component Details		ensions Iew	Cleara Nev		Remarks
	ins.	mms.	ins.	mms.	Kemarks
Thermostat. Up to Engine No. SA.1712 Valve begins to open Valve fully open Valve Lift		(75 — 80 C.) (95 C.) 7 94			Stamped No. X43570/11
Engine No. SA.17123E and future. Valve begins to open Valve fully open Valve Lift	54 — 163 F. 185 F. .312	(68° — 73° C.) (85° C.) 7.94			Stamped No. X43570/16
Oil Pump					
Approximate capacity at 50 lb/sq. in. (at engine speed).	3.52 kg/sq. c	m.) is 3.95 galls. (1	6.94 litres) per	minute at ?	2000 engine r.p.m. (Oil pump run
Oil Pressure 40-60 lb/sq. in. (2.8—4.2 l	kg/sq. cm.).				
Bore Inner Rotor	. 4987 . 4993	12.667 12.683	.0012	.030	
Pump Shaft Dia.	. 4985 . 4981	12.662 12.651	.0002	.005	
Bush Internal Día.	. 4995 . 4990	12.688 12.675	.0005	.013	
Outer Rotor Outside Dia.	I . 599 I . 598	40.615 40.589	.006	. 152	
Housing Internal Dia.	I . 604 I . 603	40.742 40.716	.004	. 102	
Rotor Depth—Inner and Outer	. 9995 . 9985	25.388 25.362			
Pump Housing Depth from flange face to bottom of bore	. 8410 . 8400	21.361 21.336			
Depth of Recess in Front Main Bearing Housing	. 157 . 156	3.988 3.962			
Gasket Thickness (Uncompressed).	.006	. 152			
End Clearance			.0035	.089	End clearance allowing for .002"
Inner Rotor Major Dia.	1.172 1.171	29.769 29.743	.0005	.013	(.051 mm.) compression of gasket. A combined worn clearance of .005" (.127 mm.)
Inner Rotor Minor Dia.	.731 .729	18.567 18.517			indicates need of facing bearing housing recess and facing.
Clearance on Rotors			.004 .001	. 102 . 025	Measured when major dia. of inner rotor and minor dia. of
Camshaft Journal Dias.	I . 5595 I . 5590	39.612 39.599	. 0045	.115	outer rotor are in line; when this clearance exceeds .010° (.254 mm.) new parts should be fitted.
Bore in Block and Front Bush Internal Dia.	1 . 5635 1 . 5620	39.713 39.675	.0025	.064	Provision made for vernier set- ting of the valve timing with camshaft chainwheel on centre.
amshaft End Float					
Front Bearing Length	I.373 I.370	34.874 34.798	.0075	. 191	
Front Journal Length	I . 3775 I . 3750	34.988 34.925	. 0020	.051	

Component Details		nsions ew	Clear Ne	rance ew	Remarks
Decans	ins.	mms.	ins.	mms.	ivemai ks
Tappets and Valves					
Bore in Block.	. 5630	14.300			
	. 5623	14.282	.0012	.030	
			.0003	.008	
Tappet Stem Dia.	. 5620 . 5618	14.275 14.270			
Valve Tip Clearance Inlet & Exhaust (Cold).			.012	.305	
Valve Guide Bore Dia.	.3130	7.950			Valve Guides not interchang
Inlet & Exhaust.	.3120	7.925	.0023	.058	able; except between Engi No. SA.17677E and SA.2940
Inlet Valve Stem Dia.	.3112	7.904	. 0008	. 020	when the inlet guide w common.
inter valve stem bla.	.3107	7.892	005	127	
			. 005 . 003	. 127 . 076	
Exhaust Valve Stem Dia.	. 309 . 308	7.849 7.823			
Valve Head Dia.					
Inlet	1 . 252 1 . 2 4 8	31.801 31.699			
5 Lucia		28.626			
Exhaust	1.127 1.123	28.524			
Valve Lift: Inlet	. 3075	7.810			
Exhaust	. 342	8.687			
Valve Seating Angle on Valve Head Valve Seat Angle in New Cylinder	1 45°. Haad 89° includes	d			When seats in Cylinder Hea
Valve Seat Aligie III New Cyllinder	riead ov include	u.			90° (incl.) cutter.
/alve Springs.	4				
Rate: Inner 56.8 lb/in. (1014.3 kg					
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5	g/m) 75 mm.) Free Le	ngth: Inner 1.51 Outer 1.6	" (38.354 mm.) 00" (40.64 mm.)	approx.	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.55 Outer 1.219" (30.96	g/m) 75 mm.) Free Le 63 mm.)	ngth : Inner 1.5 Outer 1.6	" (38.354 mm.) 00" (40.64 mm.)	approx. approx.	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5	(/m) 75 mm.) Free Le 63 mm.) 454 kg.) : kg.) Full Lift I kg.) Inne	Outer 1.6 Load : Inler 39.5 lb. (00″ (40.64 mm.) let 17.9 kg.) 41.5 l	approx. Exhaust b. (18.8 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5 Outer 1.219" (30.96 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18.	(/m) 75 mm.) Free Le 63 mm.) 454 kg.) : kg.) Full Lift	Outer 1.6 Load : Inler 39.5 lb. (00″ (40.64 mm.) et E	approx. Exhaust b. (18.8 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. /alve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (00″ (40.64 mm.) let 17.9 kg.) 41.5 l	approx. Exhaust b. (18.8 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. /alve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (00″ (40.64 mm.) let 17.9 kg.) 41.5 l	approx. Exhaust b. (18.8 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (00″ (40.64 mm.) let 17.9 kg.) 41.5 l	approx. Exhaust b. (18.8 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C Inlet opens closes 25° after B.D.C closes 25° after B.D.C	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (00″ (40.64 mm.) let 17.9 kg.) 41.5 l	approx. Exhaust b. (18.8 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5' Outer 1.219" (30.96' Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C Inlet opens 5° before T.D.C closes 25° after B.D.C closes 25° after B.D.C Closes 25° after B.D.C Closes Compression Cam Clearances. Cylinder Nos.	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (00° (40.64 mm.) let E 17.9 kg.) 41.5 l 32.9 kg.) 76.5 l	approx. Exhaust b. (18.8 kg.) b. (34.7 kg.)	
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5' Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Closes 25° after B.D.C Decompression Cam Clearances. Cylinder Nos. 1, 2 and 4	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i	approx. Exhaust b. (18.8 kg.) b. (34.7 kg.)	vertical, located, on all excep very early models, by fitting
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5' Outer 1.219" (30.96' Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C Inlet opens 5° before T.D.C closes 25° after B.D.C closes 25° after B.D.C Closes 25° after B.D.C Closes Compression Cam Clearances. Cylinder Nos.	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (00° (40.64 mm.) let E 17.9 kg.) 41.5 l 32.9 kg.) 76.5 l	approx. Exhaust b. (18.8 kg.) b. (34.7 kg.)	vertical, located, on all excep very early models, by fitting
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5' Outer 1.219" (30.96' Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. /alve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Cl	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inler 39.5 lb. (.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i	approx. Exhaust b. (18.8 kg.) b. (34.7 kg.)	vertical, located, on all excep very early models, by fitting dowel (3 da.) through 3rd pedestal extension.
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5' Outer 1.219" (30.96' Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Cl	(m) 75 mm.) Free Le 63 mm.) 454 kg.) : kg.) Full Lift I kg.) Out	Outer 1.6 Load: Inier 39.5 lb. (.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i	approx. Exhaust b. (18.8 kg.) b. (34.7 kg.)	vertical, located, on all excep very early models, by fitting dowel (1/4) dia.) through 3rd pedestal extension. Flywheels balanced individually Held to Crankshaft by 6 se
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. /alve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Decompression Cam Clearances. Cylinder Nos. 1, 2 and 4 Cylinder No. 3	g/m) 75 mm.) Free Le 63 mm.) 454 kg.): kg.) Full Lift I kg.) Out	Outer 1.6 Load : Inier 39.5 lb. (i	.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i .030 .045	.762 1.143	vertical, located, on all excep very early models, by fitting dowel (3 dowel 1
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.96) Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C clos	(m) 75 mm.) Free Le 63 mm.) 454 kg.) : kg.) Full Lift I kg.) Out	Outer 1.6 Load: Inier 39.5 lb. (.00° (40.64 mm.) let E 17.9 kg.) 41.5 l 32.9 kg.) 76.5 l	approx. Exhaust b. (18.8 kg.) b. (34.7 kg.) .762 1.143	vertical, located, on all exception very early models, by fitting dowel (1/4) dia.) through 3rd pedestal extension. Flywheels balanced individually Held to Crankshaft by 6 sescrews locked in pairs. Single
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.90 Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. /alve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Decompression Cam Clearances. Cylinder Nos. 1, 2 and 4 Cylinder No. 3	(m) 75 mm.) Free Le 63 mm.)454 kg.): kg.) Full Lift Out	Outer 1.6 Load: Inier 39.5 lb. (er 72.5 lb. (.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i .030 .045	.762 1.143	vertical, located, on all excep very early models, by fitting dowel (3/4" dia.) through 3rd pedestal extension. Flywheels balanced individually Held to Crankshaft by 6 se screws locked in pairs. Single dowel locating flywheel or
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.96) Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C Inlet opens 5° before T.D.C closes 25° after B.D.C Occompression Cam Clearances. Cylinder Nos. 1, 2 and 4 Cylinder No. 3 Flywheel. Spigot Dia. (for starter gear ring)	13.094 13.094 13.068 13.063 1.075 mm.) Free Le 1.0002	Outer 1.6 Load: Inier 39.5 lb. (er 72.5 lb. (inier 72.5 lb. (i	.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i .030 .045	.762 1.143	vertical, located, on all excepvery early models, by fitting a dowel (3/4 dia.) through 3rd pedestal extension. Flywheels balanced individually Held to Crankshaft by 6 set screws locked in pairs. Single dowel locating flywheel or
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.96) Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Cl	13.094 13.094 13.068 13.063	Outer 1.6 Load: Initer 39.5 lb. (er 72.5 lb. (for 72.5 lb.	.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i .030 .045	.762 1.143	vertical, located, on all except very early models, by fitting a dowel (18 da.) through 3rd pedestal extension. Flywheels balanced individually Held to Crankshaft by 6 set screws locked in pairs. Single dowel locating flywheel on
Outer 106 lb/in. (1892.9 kg Fitted Length: Inner 1.125" (28.5" Outer 1.219" (30.96) Fitted Load + 2 lb./—I lb. (.908— Inner 22 lb. (10 Outer 40 lb. (18. Valve Timing (Crankshaft Degrees). Exhaust opens 45° before B.D.C closes 5° after T.D.C lnlet opens 5° before T.D.C closes 25° after B.D.C Cl	13.094 13.094 13.068 13.063 1.075 mm.) Free Le 1.0002	Outer 1.6 Load: Inier 39.5 lb. (er 72.5 lb. (inier 72.5 lb. (i	.00° (40.64 mm.) let E 17.9 kg.) 41.5 i 32.9 kg.) 76.5 i .030 .045 .045	.762 1.143787584	Flywheels balanced individually. Held to Crankshaft by 6 set screws locked in pairs. Single dowel locating flywheel on

Component Details	Dimer Ne		Cleara Nev		Remarks
	ins.	mms.	ins.	mms.	
ockey Chainwheel Assembly					
Spigot Dia.	.749	19.025			
Wheel Carrier	.748	18.999	.0025	.064	
Chainwheel Bush	.7505	19.063	.0008	. 020	
Internal Bore	.7498	19.045			
Front Pivot Dia.	. 4 99	12.675			
	. 4 98	12.649	.0025	.064	
			.0008	.020	
Bore in Timing Cover	. 5005 . 4 998	12.713 12.695			
Rear Pivot Dia.	. 4998				
Rear FIVOL DIa.	. 1 776 . 4 993	12.695 12.683			
			.0012 .0000	. 030 . 000	
Bore in Cylinder Block	. 5005	12.713	.0000	.000	
	. 4998	12.695			
jector Pump Drive.					
Pump Drive	. 8748	22.220			Drive Bush for chainwheel give
Shaft Minor Dia.	.8743	22.207	.0012	.030	vernier adjustment for pum timing in $l^{\frac{1}{2}}$ stages up to 6
Locating Bush Int. Dia.	. 8755	22.238	.0000	.000	,
Locating Dash me. Dia.	. 8748	22.220			
Locating Bush Ext. Dia.	1.28105	32.538			
-	1.28055	32.526	.0014	.035	
			.0000	.000	
Chainwheel Bore	1.28195 1.28105	32.561 32.538			
Puma Daiva Shafe Majan Dia	1.1233	28.532			
Pump Drive Shaft Major Dia.	1.1233	28.486			
			.0042 .0015	. 107 . 038	
Bearing Housing	1.1257	28.593	.0013	.030	
Internal Dia.	1.1248	28.570			
rive Shaft End Float.					
Length of Shaft	2.1900	55.626			
	2.1875	55.563	.0075	. 191	
Length of Housing	2.1845	55 . 4 87	.0030	.076	
Length of Housing	2.1825	55.436			

Injector Pump and Injectors.

Injection Spill Cut-off 30 (Crankshaft) before T.D.C.

Slots in Pump Mounting Flange allows for the following movement. Pumps Pt. Nos. 300342 and 300781—total 10° (Crankshaft). Pumps Pt. Nos. 300964 and 300972—total 16° (Crankshaft).

 Injector Breaking Pressure.
 120 ats. (123.5 kg/sq. cm.).

 Spray Angle.
 4°

 Valve Lift.
 .0276 .700

Electrical Equipment.

Dynamo:

Runs at 1.72 engine speed.

Timing hole in flywheel and Cylinder Block when aligned with $\frac{1}{4}$ " (6.35 mm.) dia. tommy bar locates Nos. 1 and 4 spill cut-off timing.

Component	Dimensions		Clearance		Remarks
Details	New		New		
	ins.	mms.	ins.	mms.	

Starter Motor.

Number of Teeth in Flywheel Gear Ring and Starter Pinion 113 and 11 respectively.

End Clearance between disengaged starter pinion and Flywheel Gear Ring .090" (2.29 mm.)—Distance from mounting flange to front face of Flywheel Gear Ring 1.090" (27.69 mm.).

Starter Pilot Switch should make electrical contact when the leading face of the starter pinion is $1\frac{5}{8}$ " (41.28 mm.) from the starter motor mounting flange face.

2 — 6 volt 19 plate units connected in series. Recharge rate 12 amps.

Batteries.

Specific Gravity 1.28 — 1.30 at 60°F. (15°C.). Up to Tractor Serial No. 207705. 120 amp 120 ampere hour capacity at 10 hour discharge rate 2 — 6 volt 17 plate units connected in series. Recharge rate 13 amps. Tractor Serial No. 207706 and future. 115 ampere hour capacity at 10 hour discharge rate.

Clutch.

Clutch Dia. 254 12 green springs each of 105 lb. (47.627 kg.) to 115 lb. (52.199 kg.). Clutch Springs:

Fill-up Data.

Main. 7 Imp. Gallons (31.85 litres).
Auxiliary Tank. ½ Imp. Gallon (3.4 litres).
Ki-gass Tank. ¾ pint (.43 litres).
12 pints (6.8 litres).
¾ pint (.43 litres).
15 pints (8.5 litres). Fuel Tanks.

Engine Sump. Air Cleaner Bowl. Cooling System.

Tractor Weight Approx.:

2700 lb. (1225 kg.) 2770 lb. (1256 kg.) Up to Tractor Serial No. 325000 Tractor Serial No. 325001 and future.

PETROL ENGINE

(Manufactured by the Standard Motor Co.)

ENGINE: 85 m/m bore \times 92 m/m. stroke as fitted to Tractors Type TE-A20.

Displacement: 2088 cc. (127.4 cu. ins.)

Compression Ratio: 6:1.

Firing Order: 1, 3, 4, 2.

Maximum Belt Horse Power — 28.2.

Tightening Torques: —

Cylinder Head Nuts 60 — 65 lbs. ft. (8.29 — 8.98 kg.m).

Main Bearing Nuts

80 — 85 lbs. ft. (11.05 — 11.75 kg.m).

Big End Nuts

50 — 55 lbs. ft. (6.91 — 7.60 kg.m)

Flywheel Cap Screws 42 — 46 lbs. ft. (5.8 — 6.4 kg.m).

Component Details	Dimen Ne		Clear Ne		Remarks
	ins.	mm.	ins.	mm.	
Main Bearings.					
Housing Bore	2.6255 2.6250	66.688 66.675			For checking housing or bearing bores fit bearing cap and tighten to specified torque.
Radial thickness of Bearings.	.07225 .07200	1.8351 1.8288			·
Bearing Bore diameter	2.4815 2.4805	63.030 63.005			
(fitted).	2.4603	63.003	.0025 .0010	.064 .025	Similar tolerances for reground crank- shaft to .010", .020", .030", .040" (.254, .508, .762, 1.016 mm) undersize.
Crankshaft.					•
Journal diameter.	2.4795 2.4790	62.979 62.967			
Crankshaft End Float.					
Centre Journal length	1.7507	44.468			
	1.7498	44.445	.0117	.297	
Common Portion Com NA/Hab	1 550	30 500	.0048	.122	
Centre Bearing Cap Width	1.559 1.557	39.599 38.5 4 8			
Thrust Washer thickness (2 off)	.093 .091	2.362 2.311			
Big End.					
Crankpin diameter	· 2.0866 2.0861	53.000 52.987	.0024 .0006	.061 .015	Similar tolerances for reground crank- shaft to .010", .020", .030", .040" (.254, .408, .762, 1.016 mm) undersize.
Bearing Bore diameter	2.0885	53.048	.0000	.015	For checking big-end or bearing bores
(fitted).	2.0872	52.015		•	assemble connecting rod and tighten to specified setting.
Connecting Rod Bore diameter	2.2335 2.2327	56.731 56.710			
Radial thickness of Bearings	.07275 .07250	1.8479 1.8415			
Connecting Rod End Float					
Crankpin length	1.1915 1.1865	30.264 30.137			
			.014 .007	.356 .178	
Connecting Rod width	1.1795	29.959			
	1.1775	29.909			

iston. F. Grade G. ",	ins. 1.0000 .9995 1.0045 1.0035 .8752 .8748 .87510 .87485	25.400 25.387 25.514 25.489 22.230 22.220 22.228 22.221	*0035 0050 *0035 00030 *00045 00005	—.09 —.13 +.009 —.008	Heat piston in boiling oil for removal and fitting of gudgeon pin. *Specified clearances using drawing size but desired fit of gudgeon pin — han
iston. F. Grade	.9995 1.0045 1.0035 .8752 .8748 .87510 .87485	25.387 25.514 25.489 22.230 22.220 22.228 22.221	*0050 *00035 00030 *00045	—.13 +.009	*Specified clearances using drawing size but desired fit of gudgeon pin — han
iston. F. Grade	.9995 1.0045 1.0035 .8752 .8748 .87510 .87485	25.387 25.514 25.489 22.230 22.220 22.228 22.221	*0050 *00035 00030 *00045	—.13 +.009	*Specified clearances using drawing size but desired fit of gudgeon pin — han
iston. F. Grade	.8752 .8748 .87510 .87485	25.489 22.230 22.220 22.228 22.221	*0050 *00035 00030 *00045	—.13 +.009	*Specified clearances using drawing size but desired fit of gudgeon pin — han
iston. F. Grade	.8752 .8748 .87510 .87485	25.489 22.230 22.220 22.228 22.221	00030 *00045		*Specified clearances using drawing size but desired fit of gudgeon pin — han
F. Grade	.8748 .87510 .87485	22.228 22.221 22.233	00030 *00045		DUT desired fit of gudgeon nin han
F. Grade	.87510 .87485	22.228 22.221 22.233	00030 *00045		DUT desired fit of gudgeon nin han
F. Grade	.87485 .8753	22.221	*00045		DUT desired fit of gudgeon nin han
F. Grade	.87485 .8753	22.221			Dush fit at 68°F - obtained by selection
F. Grade				- - 011	push fit at 68°F — obtained by selective assembly.
F. Grade			00003	001	
6					
6					
_	3.3430	34.912			
G. "	3.3427	84.905			
	3.3434 3.3430	84.922 84.912			
ш					
п. "	3.3434	84.922			
•	•		.0037 .0030	.094 .076	
F. "	3.3463 3.3460				
G					Paplacament decuse surileble
- "	3.3464	84.993			Replacement sleeves available as standard size, and rebored +.020" (.508 mm)
н. "	3.3471	85.016			Oversize pistons available to suit rebored sleeves. See below.
iton	3.3 4 68	85.009	.0022	.056	Thrust side Piston Skirt tapered .0015
			.0015	.038	(.038 mm) on diameter.
	3.329 3.327	84.557 8 4 .506	.0201 .0170	.510 .432	
	.0807	2 050			Piston fitted with three rings above
	.0797	2.024	. 0030	074	gudgeon pin, one ring below.
	0707	1.000	.0010	.025	
1	.0777	1.974			
	.1895	4.813			
	.1885	4.788	.0030	.076	
lth	.1875	4.763	.0010	.025	Similar tolerances for: oversize rings
	.1865	4.737			010" (.254 mm),020" (.508 mm), 030" (.762 mm).
	.158	4.013			(.762 mm).
	.13/	3.707	.0030	.076	
lth	.156	3.962	.0010	.025	
	.155	3.937	A : -	261	
			.015 .010	.381 .25 4	
Jpper Blo	ock				
••	4.140	105.156 10 4 .775			
	==		.045 015	1.143	
iameter	4.110	104.394 104.013	.013	.100.	
	F. " G. " H. " ton	H. " 3.3438 3.3434 F. " 3.3463 3.3460 G. " 3.3467 3.3464 H. " 3.3471 3.3468 ton 3.329 3.327 .0807 .0797 .0797 .0787 .0777 .1895 .1885 th .1875 .1865 .158 .157 th .156 .155	H. , 3.3438 84.932 3.3434 84.922 F. , 3.3463 84.996 3.3460 84.983 G. , 3.3467 85.005 3.3464 84.993 H. , 3.3471 85.016 3.3468 85.009 ton 3.329 84.557 3.327 84.506 .0807 2.050 .0797 2.024 .0787 1.999 .0777 1.974 .1895 4.813 .1885 4.788 th .1875 4.783 .1885 4.788 th .1875 4.737 .158 4.013 .157 3.989 th .156 3.962 .155 3.937	H. , 3.3438 84.932 3.3434 84.922	H. , 3.3438 84.932

Component Details	Dimensions New		Clearance New		Remarks	
	ins.	mm.	ins.	mm.		
Clearance between :— Sleeve Spigot and Lower Bloo Lower Block diameter	:k 3.6260	92.100				
	3.6245	92.062	.003 .0005	.076 .013		
Sleeve Spigot diameter	3.6240 3.6230	92.049 92.024		.515		
Cylinder Block (Top face to seating face for Sleeve)	4.501 4.499	114.325 114.274				
Gasket thickness (uncompressed)	.019 .016	.483 .406				
Cylinder Sleeve (Top face to seating face).	4.488 4.487	113.995 113.970				
Stand out of Sleeve above Cylinde Block.	er		.002 .008	.051 .203		
Oversize Sleeves & Pistons Piston Diameter	3.3632	85. 4 25				
+.020" (.508 mm).	3.3627	85.413	00.43			
			.0043 .0030	.110 .076		
Sleeve Bore (Parallel) : .020" (.508 mm).	3.36700 3.36625	85.522 85.503				
Water Pump. Housing Bore for Bearings	1.5749	40.002				
	1.5744	39.990	+.00061	+.015		
			- .00040	—.010		
Bearing Case external diameter	1.57480 1.57429	40.000 39.987				
Spindle diameter	.6262 .6257	15.905 15.893				
			—.0032 —.0022	081 056	Drive to impellor imparted throu- interference fit of impellor on spind	
Impellor Bore diameter	.6235 .6230	15.837 15.824	0022	030	Incorporated at Engine No. SC.41539	
hermostat.						
Valve begins to open Valve fully open Valve lift	133° — 14 171° F. ‡" — }"	47°F.	56` — 64°C 77 C. 6.3 — 9.5 r			
runs at half engine speed).			.95 gallons (17.9	'S litres) per	minute at 2,000 r.p.m. Oil Pump (Oil Pun	
Oil pressure 40 — 60 lbs./sq. in. (3	2.8 — 4.2 k _i 1.599	g. sq. cm.). 40.615				
Outer Rotor, outside diameter	1.598	40.589				
			.006 .004	.152 .101		
Housing Internal diameter	1.604 1.603	40.741 40.716				
Bore, Inner Rotor	. 4 987 . 4 993	12.667 12.682				
	. 7773	12.002	.0002	.005		
Pump Shaft diameter	. 4 980	12.649	.0013	.033		
	.4985	12.661	002	.076		
			.003 .001	.025		
Housing Bore internal diameter	. 5 010 . 4 995	12.725 12.687				
nd Clearance of Rotors. Rotor depth — outer and inner	0.9995	25.387				
Notor deptil — Outer and liller	0.9985	25.362				
			.0005 .0025	.013 .06 4	A combined worn clearance of .00 (.127 mm) indicates need of cover as	
Housing depth	1.001	25.425	.5025	.50 1	housing face lapping.	
	1.000	25.400				

Component Details	Dimensions New		Clearance New		Remarks
Inner rotor, major diameter	ins. 1.171 1.172	mm. 29.743 29.769	ins.	mm.	1
Inner rotor, minor diameter	.729 .731	18.517 18.567			
Clearance on rotors	., 3,	10.307	.004 .001	.102 .025	Measured when major diameter of inner rotor and minor diameter of outer rotor are in line. Where clearance exceeds .010" (.254 mm) new parts should be fitted.
amshaft.					
Front Journal diameter	2.0595 2.0590	52.311 52.299	.0045 .0025	.114 .063	Max. wear on camshaft journals .003' (.076 mm) and .0035" (.089 mm) ir
Bore in Block.	2.0635 2.0620	52.413 52.375			cylinder block.
2nd Journal diameter 3rd ,, ,, }	1.71575	43.580			
Rear ,, ,,	1.71525	43.567	.00445	.113	
Bore in Block	1.7197 1.7182	43.680 43.642	.00245	.062	
amshaft End Float. Locating Groove	.1885 .1865	4.788 4.737	.0065	.165	
Locating Plate	.1835 .1820	4.661 4.623	.0030	.076	
ippets and Valves					
Tappet Bore in Block	.9380 .9373	23·825 23·807	.0013	.033	
Tappet diameter	.9371 .9367	23.802 23.792	.0002	.005	
Valve Tip clearances (cold) Inlet			.010	.254	·
Exhaust			.012	.305	
Inlet valve stem diameter	.311 .310	7.899 7.874	001	025	
Valve Guide Bore diameter (Inlet and Exhaust)	.313 .312	7.950 7.925	.001 .003	.025 .076	
(miet and Exhaust)	.312	7.723	.003	.076	
Exhaust Valve Stem diameter	.309 .308	7.849 7.823	.005	.127	
Valve Head diameter : Inlet	1.113 1.109	28.27 28.17			
Exhaust	1.051 1.047	26.695 26.594			
Guide projection above spring seat	.59	15.0			
Valve lift	.2985	7.582			4

Valve seating angle on valve head 45 . Valve seat angle in new cylinder head $44\frac{1}{2}$, but serviced at 45° Exhaust Valve inserts available for service purposes:

Valve Springs.

Free length 1.787" (45.390 mm). Fitted load 38 lbs. — 2 lbs. (17.237 kg. +.907 kg.) — .454 kg.) Fitted length 1.321" (33.553 mm). Full lift load 60 lb. (27 kg.) approx.

Component Details		Dimensions New		Clearance New		Remarks	
		ins.	mm.	ins.	mm.		
Valve Timing (C	Crankshaft Degrees).						
		Exhaust o	pens 40° befo	re B.D.C. close	s at T.D.C.	Timing hole in the flywheel and crank-	
		Inlet open	s at T.D.C. c	loses 40° after	B.D.C.	case when aligned with tommy bar locate No. I and No. 4 T.D.C.	
Flywheel.							
	er (for Starter	13.406	340.512				
Gear Ring	g).	13.403	340.436	031	—.787		
Starter Gear P	ling (inside diameter)	13.380	339.852	— .023	—.584		
		13.375	339.725				
Crankshaft Sp	igot diameter	4.0007 4.0002	101.617 101.605				
		4.0002	101.003	0012	÷.0305		
Flywheel diam	eter for Spigot.	3.9998	101.595	—.000 4	—.0102		
		3.9995	101.587				
Run out of Clu	utch contact face at o	uter diame	ter should no	t exceed .003"	(.076 mm).		
Carburetter.							
Zenith type 24 (l T — 2 (fully variabl Choke tube	e main jet). 17					
	1ain jet Iow running jet	100 60			Inter C	Con. I mm. drilled.	
P	rogression	120 (2 holes) Air jet				2.0	
	leedle seating 'etrol level at 4′6″ he	2.0 m ad 15 m			Carbui	retter bowl stamped "P".	
Battery:	Tractor Serial N 1.28 — 1.30 full	No. 200,001 s y charged as	and future — t 60°F. (16°C.)	12 volt 38 amp	. hour capacit	y at 10 hour discharge rate. Specific Gravity	
Distributor :	Static Setting I	° (Cranksha	ıft) before T.I	D.C. Co	ontact Breake	r Gap .014" — .016" (.36 — .41 mm.)	
Spark Plug.							
Reach		.5	12.7				
Thread diameto Gap	er	.032	mm. .81				
		.030	.76				
Starter Motor :	No. of Teeth : Distance of from	•	_			• •	
Dynamo :	Runs at 1.9 × 0	engine spee	d.				
Governor:	Range 400 — 2	000 r.p.m. (Engine).				
	Governor Leve				001	5// / 127	
		_		" (96.5 mm). E	nd Play .00	5" (.127 mm). 0" (.254 mm) Rate 20 lbs/in. ± 5%.	
		Coils 243. wound-in lo		.5.4 mm) deflec	tion: 20 lbs	(9.07 kg.) + 2 lb. (.907 kg.).	
	Control Rod Co	ompensating	g Spring : ide hooks 2.1	87" (55.55 mm)	Rate 64 lbs	i/in 5%. No. of coils 11½.	
	Load a	t ½" (12.7 m	m) deflection oad : 6 lbs. (2	: 38 lbs. (17.2	237 kg.) 🚠 1½	lbs. (.681 kg.).	
ill-up Data:							
		Imperial gal pints (6.8 l		s), including 1 l	mperial gallo	n (4.5 litres) reserve.	
Air	Cleaner Bowl 🔒	pint (0.43 li	tres)				
_	ring Gear-box 5	pints (2.9 lit	lons (22.8 litr tres)	es)			
3166							
Fron		pint (0.35 li: pint (0.28 li:	tres)				

VAPORISING OIL ENGINE

(Manufactured by the Standard Motor Co.)

This information supersedes that originally issued on Pages B.9 to B.12.

Engine: 85 mm bore \times 92 mm stroke as fitted to Tractors Type TE-D 20.

Displacement: 2088 CC (127.4 cu. ins.)

Firing Order: 1, 3, 4, 2.

There are several types of the V.O. Engine, namely :-

Engine Nos. S120,510E to S170,173E covering versions I & 2.

Compression Ratio: 4.8 to I.

Max. Belt H.P.: 23.9

Engine Nos. S170,174E onwards, covering versions 3 & 4.

Compression Ratio: 5.1: 1.

Max. Belt H.P.: 25.4

Component Details		Dimensions New			Clearance New		Remarks
			ins.	mm.	ins.	mm.	
Pistons & Sleeves.							
Piston Dia. (Thrust Side Top Skir		irade	3.3430 3.3427	84.912 84.905			Piston fitted with four rings above
	G.	,,	3.3434 3.3430	84.922 84.912			gudgeon pin, one ring below. These grades are identical with those of 85 m/m bore petrol engine.
	Н.	••	3.3438 3.3434	84.932 84.922	0027	004	Oversize pistons available to suit rebored sleeves.
					.0037 .0030	.094 .076	
Sleeve Bore (Parallel)	F.	,,	3.3463 3.3460	84.996 84.988			Sleeves identical with those of 85 mm bore petrol engine.
	G.	,,	3.3467 3.3464	85.006 84.998			Replacement sleeves available as standard size, and rebored + .020" (.508 mm)
	Н.	,,	3.3471 3.3468	85.016 85.009			(======================================
Clearance, Bottom of Skirt (Thrust Side).	Piston				.0022 .0015	.056 .038	Thrust side Piston Skirt tapered .0015' (.038 mm) on diameter.
Top Land Diameter			3.329 3.327	84.557 84.506	.0190 .0170	.483 .432	
Ring Groove Width Top, 2nd, and 3rd.			.0807 .0797	2.050 2.024	.003 .001	.076 .025	
Compression Ring Wi Top, 2nd, and 3rd.	idth.		.0787 .0777	1.999 1.974			
Ring Groove Width 4th			.1895 .1885	4.813 4.788			
Slotted Scraper Ring \ 4th	Width		.1875 .1865	4.763 4.737	.003 .001	.076 .025	Similar tolerances for oversize ring010" (.245 mm),020" (.508 mm030" (.762 mm).
Ring Groove Width 5th			.158 .157	4.013 3.987			
					.003 .001	.076 .025	
Slotted Scraper Ring ' 5th	Width		.156 .155	3.962 3.937			
Ring Gap (closed)					.015 .010	.381 .254	

Component Details	Dimen Ne		Clear Ne		Remarks	
	ins.	mm.	ins.	mm.		
Oversize Sleeves & Pistons Piston Diameter 020" (.508 mm).	3.3632 3.3627	85.425 85.413	.0043	.110	; ;	
Sleeve Bore (Parallel) + .020" (.508 mm)	3.36700 3.36625	85.522 85.503	.0030	.076		
Valve Head Diameters.						
Version I. Engine Nos. SI2 Inlet	0,510E — S121, 1.252 1.248	954E. 31.800 31.699				
Exhaust	1.252 1.248	31.800 31.699				
Version 2. Engine Nos. SI2 Inlet	1, 955E — S170, 1.176 1.172	1 73E. 29.870 29.769				
Exhaust	1.051 1.047	26.695 26.594				
Version 3. Engine Nos. S17	0,174E — S225,	311 E .				
Inlet	1.113	28.270 28.169				
Exhaust	1.051 1.047	26.695 26.594				
Version 4. Engine Nos. S22	5,312E onward : 1.238 1.234	s. 31.445 31.344				
Exhaust	1.051 1.047	26.695 26.594				
Thermostat.						
Valve begins to open Valve fully open Valve lift	167 F—176 203 F. .312	5°F 75°C− 95°C. 7.9 4	–80°C.			
Carburettor.						
Zenith Type 24 T-2 (fully variat	Choke Tub Main Jet Slow Runn Progression Needle Sea Fuel level a Inter-con Air Jet	ing Jet 1	ped	17 105 60 120 (2 holes) 2.0 mm. 15 mm. 1 mm drilled. 2.0 mm. "V.O."		
Distributor. Static Setting 4 (c	rankshaft), befo	re T.D.C. (V	ersions 1 & 2	and early editions of ve	rsion 3, 6° (Crankshaft).	
Spark Plug						
Reach Thread diameter Gap	.75 14 :	19.05 mm, .89				
	.030	.76				
Fill-up Data.				_		
Fuel Tank. Vaporisin	ng Oil: 7 Impe	erial Gallons	(31.5 litres)	Petrol: I Imp	erial Gallon (4.5 litres).	

LAMP OIL ENGINE

(Manufactured by the Standard Motor Co.)

ENGINE: 85 mm bore \times 92 mm stroke as fitted to Tractors Type TE-H 20.

Displacement: 2088 CC (127.4 cu. ins.)

Firing Order: 1, 3, 4, 2. Compression Ratio: 4.5:1.

Maximum Belt H.P.: 22.9

Component Details		Dimensions New		Clearance New		Remarks
		ins.	mm.	ins.	mm.	
Pistons and Sleeves.						
Piston Diameter. (Thrust Side Top Skirt)	F. Grade	3.3435 3.3432	84.925 84.917			Piston fitted with four rings abov gudgeon pin, one ring below.
	G. "	3.3439 3.3436	84.935 84.917			Oversize pistons available to suit rebore
	н. "	3.3443 3.3440	84.945 84.938			sleeves.
			, 01.730	.0031 .0025	.079 .064	
Sleeve Bore (Parallel)	F. "	3.3463 3.3460	84.996 84.988			Sleeves identical with those of 85 m/m bore petrol engine.
	G. "	3.3467 3.3464	85.006 84.998			Replacement sleeves available as standard
	н. "	3.3471 3.3468	85.016 85.009			size, and rebored — .020" (.508 mm).
Clearance, Bottom of Pist Skirt (Thrust side)	on			.0016 .0010	.041 .025	Thrust side Piston Skirt tapered .0015" (.038 mm) on diameter.
Top Land Dia.		3.328 3.325	84.531 84.455	.0221 .0180	.561 . 4 57	
Ring Groove Width Top, 2nd and 3rd		.0817 .0807	2.075 2.050			
		.0007	2.030	.004	.102	Similar tolerances for oversize rings
Compression Ring Width Top, 2nd and 3rd		.0787 .0777	1.999 1.974	.002	.051	010" (.245 mm)020" (.508 mm) 030" (.762 mm).
Ring Groove Width 4th and 5th		.1905 .1895	4.839 4.813			
				.0040	.102	
Slotted Scraper Ring Widt 4th and 5th	:h	.1875 .1865	4.763 4.737	.0020	.051	
Ring Gap—closed				.015 .010	.381 .25 4	
versize Sleeves & Pistons	;					
Piston Diameter :: .020" (.508 mm).		3.3637 3.3632	85.438 85.425	0020	007	
CI				.0038 .0025	.096 .064	
Sleeve Bore (Parallel) 020" (.508 mm).		3.36700 3.36625	85.522 85.503			
park Plug						
Reach Thread diameter		.75	19.05			
Gap		.032	mm. .81			
•		.028	.71			

The Thermostat and Carburettor are identical to the vaporising oil engine. See page B28.

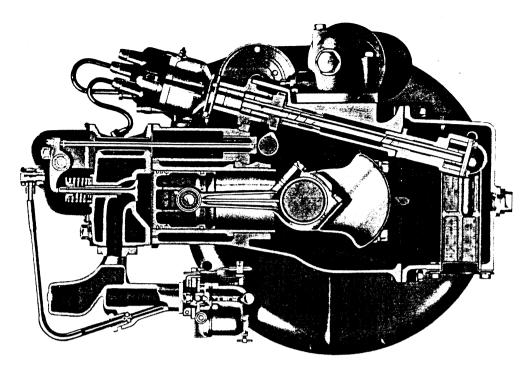
Fill-up Data.

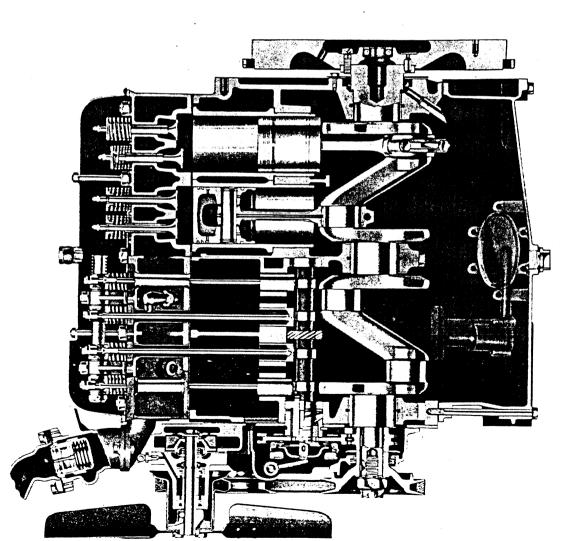
Fuel Tank. Cooling System:

Lamp Oil. 7 Imperial Gallons (31.5 litres). 17 pints (9.6 litres.)

Petrol I Imperial Gallon (4.5 litres).

For other details and data — Refer to 85 mm bore petrol engine. Pages B21 to B25.





SECTIONAL VIEWS OF PETROL ENGINE

ENGINE SECTION

The following instructions assume the use of the specially designed dismantling stand and range of tools and equipment listed in Section U of this Manual.

Separate dismantling and assembly instructions have been prepared for the removal and replacement of all main components, but those covering replacement of the crankshaft assume a complete engine overhaul.

The comprehensive dimensional specification, presented in Section B, should be referred to when inspecting components for wear. Screw threads should always be lubricated before tightening.

Petrol engine as fitted to Tractors Type TE-A2O and TE-C2O

Lubrication System

It is proposed under this heading to describe the passage of oil through the engine, and to explain the function and dismantling procedure for the various components connected with oil distribution (i.e. oil filter and oil pump).

Oil Distribution (Fig. 1).

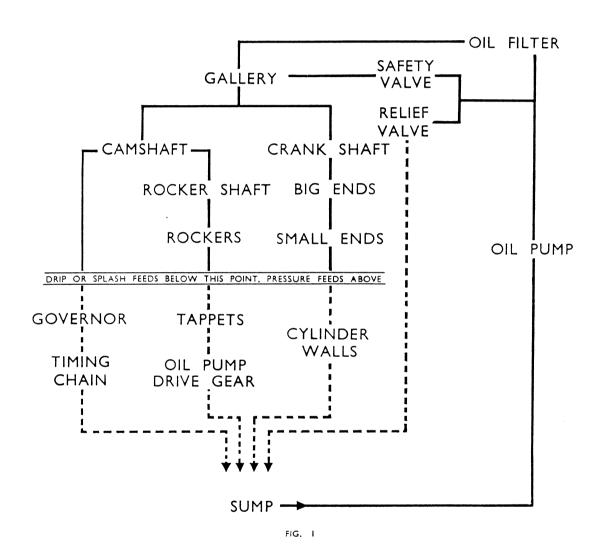
Oil is pumped from a wire mesh screen through an annular space around the pump drive shaft to the filter, from which it passes to the oil gallery for distribution as indicated in the table overleaf. Oil leakage between shaft and upper bush is prevented by a reverse spiral groove on the upper portion of the shaft.

Front, centre and rear journals of both crankshaft and camshaft are connected by drillings directly to the oil gallery, whilst

oil arrested by rear crankshaft seal drains through rear main bearing cap to sump. Lubrication of connecting rod big-end bearings is provided through drillings in crankshaft connecting front journal and No. I crankpin, centre journal and Nos. 2 and 3 crankpins and rear journal and No. 4 crankpin. The clearance between connecting rod bearing and crankpin provides sufficient flow of oil under pressure to pass along a central drilling in connecting rod to gudgeon pin, with a further surge when the connecting rod drilling aligns the supply drilling at the crankpin. Cylinder walls are lubricated by intermittent splash feed through a bleed hole drilled in the connecting rod.

A spiral oil groove connects two flats which are machined on the camshaft rear journal in such positions that as the shaft revolves,

one approaches the oil inlet drilling from the gallery, while the other approaches a second drilling through the cylinder block and head to the rear rocker shaft pedestal. Momentarily, during each revolution of the from rockers on to adjusters and push rods, returning by gravity through tappet chambers to sump. This drip feed also helps to lubricate the intermediate camshaft journal, while excess oil from rocker gear is drained



camshaft, both holes are uncovered by the flats and oil under gallery pressure passes along the spiral groove and up to the hollow rocker shaft where it is distributed through drillings to internally grooved rocker bushes. Oil is directed through relief holes

from cylinder head through crankcase to sump.

NOTE: Valve guide tops stand proud of cylinder head to avoid possibility of oil drainage to combustion chambers.

Lubrication of timing chain and governor mechanism is provided through a drilling from the circular groove in the front camshaft journal to a slot on the timing sprocket mounting face, from which oil is dispersed by centrifugal force, to drain to sump through a recess on forward face of front main bearing cap.

The Oil Pump (Fig. 2)

After detachment, the oil pump can be dismantled by removal of 4 set screws securing cover assembly to pump body, revealing the spindle and inner rotor as-

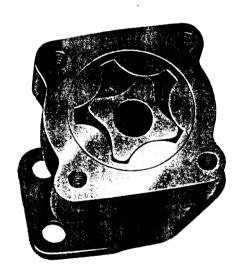


FIG. 2.

sembly which drives the outer rotor. It will be seen from Fig. 2 that the outer rotor has 5 lobes while the eccentrically mounted inner rotor has four. Thus, as the spindle turns, oil from the vacant space over the inlet port is transferred to the outer port, from which it is expelled as the lobes engage. The pump requires no attention other than washing in paraffin during a major engine overhaul.

The Oil Filter: (Fig. 3)

Vertical type X—fitted to tractors type TE-A20 before Serial No. 56340 :

Inclined type Y—fitted to tractors type TE-A20 after Serial No. 56339:

Although the above oil filters are different in appearance, they have the same principle of operation and both house a pressure relief and safety valve. Oil is pumped from the sump through port A to the outside of the filter element. Particles of dirt are removed as the oil passes through to the inside of the element and escapes to the gallery through diagonal port B. If tightness of bearings or low viscocity of oil causes the pressure in the system to rise above 40—60 lbs. sq. inch (2.8-4.2 kg. per sq. cm.), unfiltered oil passes back to the sump as pressure relief valve C opens. Should the element become clogged and the passage of oil severely restricted, oil in the diagonal port B and gallery will be at lower pressure

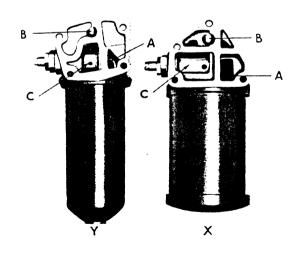


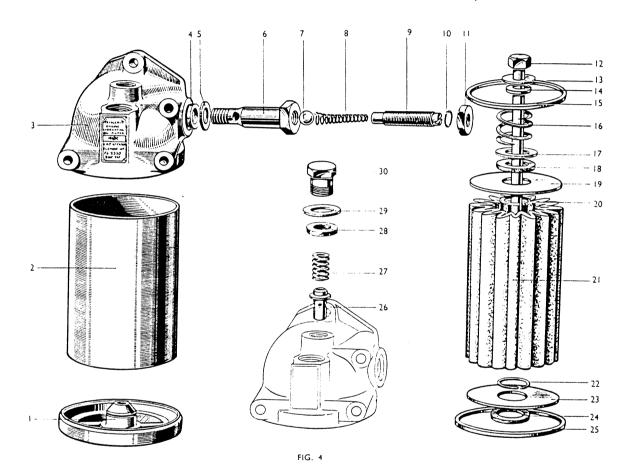
FIG. 3.

than that on the outside of the element, opening a safety valve at the lower end of port B and allowing unfiltered oil to pass into gallery.

To Dismantle Vertical Type (see Fig. 4)

- I. Remove long bolt 12 which screws into boss filter base.
- 2. Tap head assembly 3 free from barrel 2, withdraw element 21 and remove 'C' spring 20 from boss.
- 3. Remove top pressure plate 19, felt washer 18, plain washer 17, and element locating spring 16.

- 7. Tap base assembly off barrel 2.
- 8. From base assembly remove 'C' spring 22. Detach pressure plate 23 and felt washer 24 from boss and rubber jointing 25 from its groove.
- 9. Wash all parts in paraffin and rebuild base assembly.



Brown Carlot

- 4. Unscrew and remove safety valve plug 30, spring 27, and valve 26. Remove plug washer 29, and rubber sealing washer 28 from recess in plug boss.
- 5. Withdraw pressure relief valve assembly with washer 5 and seal 4 after unscrewing the larger hexagon on body 6.
- 6. Dismantle pressure relief valve assembly by unscrewing locknut II and plug 9 and withdrawing spring 8 and ball 7.

Re-Assembly of vertical type

Before re-assembly, carefully scrape all paint and dirt from machined faces and bosses, wash all parts in paraffin and blow out with compressed air. Re-new any defective part, paying particular attention to rubber and felt sealing washers.

1. Insert ball 7 in pressure relief valve body 6 and locate spring 8 with its

wide end on ball. Screw adjustment plug 9 into body and place over its slotted end a new lead packing ring 10 before screwing down locknut 11. As a preliminary setting, the plug should be screwed in so that about 6 — 7 threads are exposed above lock nut.

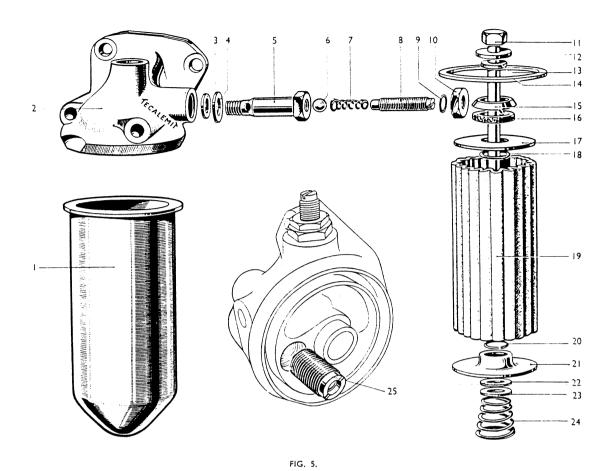
- 2. Place rubber sealing washer 4 into recess and screw down the valve assembly on to washer 5.
- 3. Place safety valve 26 in its bore and locate spring 27 on valve shoulder.
- Locate rubber sealing washer 28 in its recess and tighten plug 30 on to its washer
- 5. Place spring 16, steel washer 17, felt sealing washer 18, and pressure plate 19 over central boss inside filter head, and secure by fitting 'C' spring 20 in groove.
- 6. Replace joint ring 25 in groove of base, and secure felt washer 24 and pressure plate 23 by 'C' spring 22.
- 7. Assemble base, head and barrel with new filter element 21 and secure by screwing down long bolt 12 on to washer 13, having inserted rubber seal 14.

Inclined Type-General (Fig. 5)

This filter can be mounted in place of the vertical type without the necessity of drilling and tapping further bolt holes in the engine crank case. The filter further differs in that the barrel and base are cast in one unit to form the container. The safety valve also is of different design. Although instructions are given for removal and replacement of bottom pressure plate 21, the operations are awkward, and not usually necessary.

To Dismantle inclined type.

- 1. Remove long bolt 11 which screws into a boss in the base of container 1. Tap off head 2 and remove sealing washer 13, plain washer 12, and ring 14.
- 2. From filter head remove "C" spring 18, clamping plate 17, felt washer 16, and steel dished washer 15.
- 3. The safety valve assembly 25 can now be unscrewed and removed for cleaning, but not further dismantled.
- 4. Instructions relative to pressure relief valve are identical with those for vertical type filters as these valves are interchangeable.
- 5. From screwed boss in the base of container, remove "C" spring 20 to release pressure plate 21, felt washer 22, steel washer 23, and pressure plate spring 24.



Re-Assembly of inclined type.

Clean all parts as recommended for vertical type filters, then proceed as follows:—

- 1. Insert ball 6 in pressure relief valve body 5 and locate spring 7 with its wide end on ball. Screw adjustment plug 8 into body and place over its slotted end a new lead packing ring 9 before screwing down lock nut 10. As a preliminary setting, the plug should be screwed in so that about 6—7 threads are exposed above lock nut. Place rubber sealing washer 3 into its recess and screw down the valve assembly on to washer 4.
- 2. Replace safety valve assembly 25.

- 3. On filter head 2, replace steel dished washer 15, felt washer 16 and clamping plate 17. Secure by fitting "C" spring 18 in its groove.
- 4. In base of container, replace pressure plate spring 24, steel washer 23, felt washer 22, and pressure plate 21. Secure by fitting "C" spring in its groove.
- Replace joint ring 14 in its recess in head, locate head on container and replace sealing washer 13. Secure by tightening long bolt 11 on to its washer 12.

OVERHAUL OF CYLINDER HEAD ASSEMBLY

Necessity for Decarbonisation and Valve Grinding.

The combustion of the fuel-air mixture supplied to an engine cylinder should spread rapidly and evenly from the spark plug points through the charge. The heat of combustion should rapidly expand the gases produced by the combustion, and exert a steady sustained pressure on the piston.

The gradual build up of carbon deposits from the burnt gases inside the combustion chamber eventually adversely affects combustion because the deposits become incandescent from the heat of combustion, and prematurely ignite the incoming gas charge. Also, they cause areas of irregular combustion, giving sharp, violent increases of pressure on the piston.

The actual increase in compression ratio and poor heat dissipation due to layers of carbon can be sufficient to cause the very high pressure and temperature necessary for spontaneous combustion of fuel well before the piston has reached the top of its stroke.

This inefficient combustion is usually made evident by a characteristic metallic knocking or 'pinking,' or by a tendency for the engine to 'run on' after the ignition is switched off.

Loss of Compression.

Starting difficulty, loss of power, or increased fuel consumption will often indicate that cylinder compression is being impaired by carbon deposits in the valve guides which prevent valves from closing properly,

or valve seating faces having become pitted or burnt and which need regrinding, re-cutting or even replacing.

Testing Cylinder Head Compression.

Cylinder compression can be tested by either of the following methods:—

- (a) By turning crankshaft two complete turns with starting handle, a comparison can be made of the compression resistance of the four cylinders and thus the condition of the four pairs of valves.
- (b) By connecting a pressure gauge with suitable adaptor to each of the spark plug holes in turn and operating the starter with ignition switched off. It is important for this test that the battery is fully charged. At cranking speed the cylinder-head pressure of an engine in good condition should be 90 to 100 lbs. per square inch. (6.33—7.03 kg. per sq. cm.)

The important point to note during both these tests is any deficiency in compression for one particular cylinder compared with the average for the other three.

Visual Indication of Need for Decarbonisation.

Decarbonisation is usually necessary after about 300 hours work. An indication of the condition of combustion chamber can be obtained from that of the spark plugs. If, on removal for examination, the plugs have acquired a considerable amount of carbon around the base of the electrodes, a similar state can be assumed to exist in the combustion chambers.

The following instructions are necessary for the complete reconditioning of the cylinder head. It is unlikely that the complete overhaul will be necessary until the engine has done many hundreds of hours work, but removal of carbon deposits and lapping of valves should be carried out at regular intervals.

Preparations for Lifting Cylinder Head (Fig. 6).

- I. Remove radiator filler cap and drain water from radiator and cylinder block by opening the two drain taps, one of which is located at the base of the radiator, and the other on cylinder block behind dynamo.
- 2. Detach hood. This entails removal of two support attachment bolts from forward end of fuel tank, and two shoulder screws from radiator support bracket.
- 3. Remove hood.
- 4. Turn off fuel, disconnect fuel pipe at sediment bowl, and remove tank, which is secured by four bolts.
- Remove crankcase breather pipe by unscrewing the banjo connection at rocker cover and screwed adaptor at manifold.
- 6. Remove cotter pin from governor spring link lever I at forward end of throttle rod 2. Pull off link and allow to remain suspended on spring 3.
- 7. Disconnect radiator stay and hose 4 at water outlet elbow 5.
- 8. Disconnect by-pass hose 6 at thermostat body 7 or water pump 8 and remove the two set screws securing thermostat body to flange at forward end of cylinder head.
- 9. Swing clear the assembly comprising thermostat body and water outlet elbow, pivoting on throttle rod, as shown in Fig. 6.

- Disconnect exhaust pipe clip at engine mounting flange.
- Using service wrench FTB2, remove manifold nuts. Pull manifold off its studs and leave suspended clear of cylinder head as shown in Fig. 6. Remove gaskets.
- 12. Remove rocker cover with gasket.

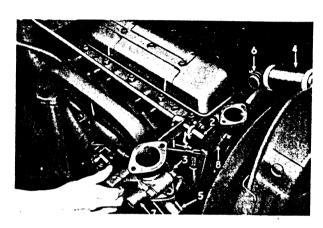


FIG. 6

- 13. Lift rocker shaft assembly after removing the four pedestal securing nuts from their studs.
- 14. Lift out push rods ensuring that tappets are not disturbed.
- 15. Remove spark plugs and immerse in petrol.

To Remove and Dismantle Cylinder Head Assembly.

- 1. Remove the ten cylinder head securing nuts from their studs.
- 2. Pulling at thermostat flange at forward end, and rocker cover flange at rear, remove cylinder head and gasket.

Note:-

Do not break the seal between head and block by turning engine. This may disturb sleeves.

3. On to the second and fourth cylinder head studs on manifold side, screw sleeve retainers—FT.3, Fig. 7.

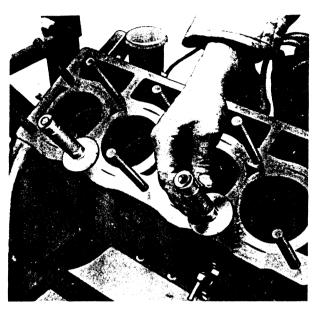


FIG. 7

4. Mount cylinder head assembly on service fixture FTB9, with pegs located in end stud holes. Adjust the position of the strut to obtain a vertical pull on collar. Compress valve springs and remove split cones. Valves and springs should be placed in order of removal on stand as shown in Fig. 8.

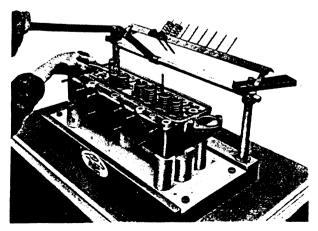


FIG. 8

Note:-

If oil retaining cups and seals are fitted above valve guides, they should be discarded.

Replacement of Push Rod Tubes:-

Damaged push rod tubes can easily be replaced using service tool FT.53, Fig. 9.

To Remove Tube.

- I. Cut through the tube with a hack-saw. Fig. 10 inset.
- 2. Knock out each half of the tube from its location in cylinder head.

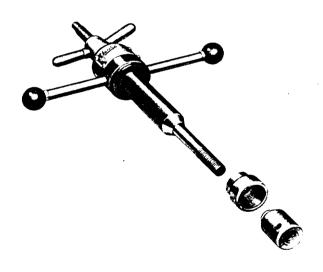


FIG. 9

To Fit New Tube.

Place collar A on tool shank as shown in Fig. 10.

- 2. Locate new tube in cylinder head flange.
- 3. Insert tool shank through tube and tighten tapered collar B.
- 4. Turn handle C to pull tube into position.

- 5. Unscrew and remove tool.
- 6. Replace tool without collar A, secure tapered collar B. and tighten by turning handle C. to locate taper of tool body in tube.

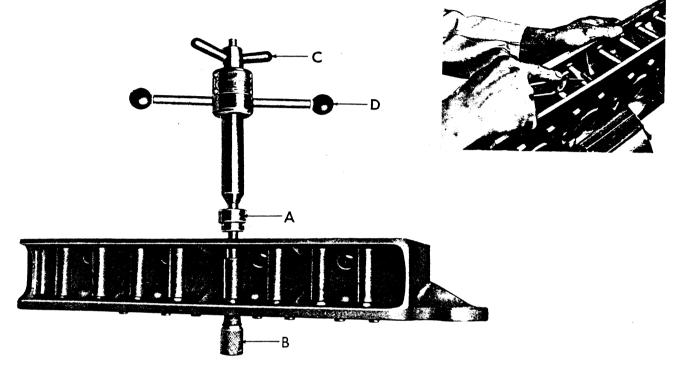


FIG. 10

- 7. Turn handle lever D to spin out edge of tube.
- 8. Continue tightening and spinning until edge of tube is spread into recess to give firm location.

Rocker Assembly (Fig. 11)

Pedestals I, rockers 2 and 3, and springs 4 can be slid off the rocker shaft 5 after removal of one of the end collars 6 and the shaft locating plug 7 from No. 4 pedestal. Prior to tapping off the end collar, it is necessary to tap out its locating pin 8.

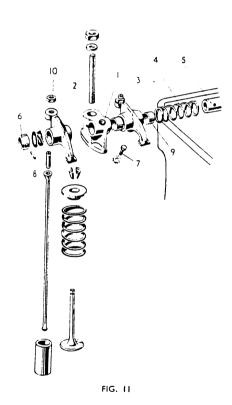
Should any of the rocker bushes 9 have worn, the assembly comprising rocker and bush must be renewed. However, adjusting pins and locknuts 10 are available as spare parts.

After blowing out all oil drillings with compressed air, re-assemble as follows:—

- I. Assemble rockers and springs on shaft as shown in Fig. II with pairs of rockers converging towards valve pad ends and with the rear pedestal, which has the oil feed drilling, located at the radially drilled and tapped shaft end.
- 2. Tap on end collar, insert pin and peen over.
- Position shaft in the drilled pedestal so that locating plug can be inserted and tightened over its shake-proof washer.

Tappets and Pushrods.

Examine each tappet in turn. Face markings illustrated in Fig. 12A., indicate that the tappet has not been turning in its bore, while those illustrated in Fig. 12B indicate satisfactory turning, thus ensuring even wear and satisfactory rocker adjustment. Tappets which have not been turned should either be replaced or relocated in another bore where satisfactory rotation has taken place. If a push rod is bent or has worn seating, it should be replaced.



Removal of Carbon Deposits.

An examination of carbon deposits on the cylinder head, combustion chambers and piston crowns will indicate the general mechanical conditions of the engine. A hard dry deposit shows that the piston rings etc., are not unduly worn. The reverse is indicated if the carbon is soft and oily.

Pistons and Sleeves.

- 1. Turn crankshaft until two pistons are about $\frac{1}{4}$ " (6.35 mm) before the top of their stroke and insert an old piston ring in the bore on top of one of them.
- 2. Place sufficient clean rag in remaining cylinders to ensure that carbon scrapings do not enter the bores.
- 3. Using a suitable scraper, carefully remove from the piston crowns all carbon inside the old piston ring, leaving a ridge around the piston edge and at the top of the sleeve bore. This helps to retain compression.

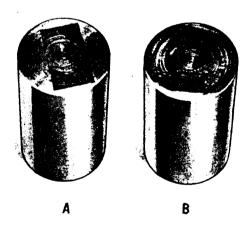


FIG. 12

- 4. Repeat this procedure for the remaining three pistons, taking care that no scrapings or chips drop between the sleeves into the cylinder block water jacket. Wipe off piston crowns with petrol-moistened rag.
- 5. Scrape any particles of dirt or grit from the upper machined surface of cylinder block and wipe off with a petrol-moistened rag.

Note:-

To ensure that no loose carbon particles remain around piston edges, thin oil can be squirted on to the edges of piston crowns, and the crankshaft rotated. Any loose particles will be left on the bores, and can easily be wiped away.

Cylinder Head.

- Examine water jacket. Immerse head in caustic soda solution if necessary to clear scale. Dismantle and clean thermostat assembly—see Section D.
- 2. Remove all carbon deposits from combustion chambers, valve ports and guide shrouds by means of suitable scrapers, or pneumatic hand tool, as shown in Fig. 13.

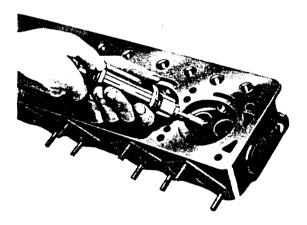


FIG. 13

3. Thoroughly clean all machined faces and wash in petrol or paraffin.

Valves, Seats and Guides.

I. Thoroughly scrape all carbon from each valve head, wash valve in petrol, afterwards polishing head and stem, preferably using a rotating wire buffer similar to that illustrated in Fig. 14.

Be careful to replace each valve in its correct position on the Service Fixture so that it can be subsequently fitted in its correct guide.

2. When each valve is thoroughly cleaned, examine the fit in its guide bore.

The clearance between stem and guide should be:—

3. If the clearance appears excessive, the valve stem diameter should be measured. The diameter of unworn stems should be:—

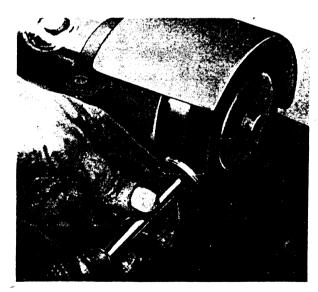


FIG. 14

After this examination, and before further treatment, it should be decided whether wear on valve stems and guides warrants replacement.

Spark Plugs.

After plugs have been washed in petrol and allowed to dry, the insulators and electrodes should be cleaned with a wire brush, or, preferably, with one of the proprietary spark plug service units. Remove all traces of grit or carbon from screw threads. Examine the insulators for cracks and the electrodes for signs of excessive burning; if damage of this nature is suspected, plugs should be renewed. If it is decided that a plug is worthy of further use, the centre and side electrodes should be dressed with a small smooth file, and the gap set to

0.028 / 0.082 ins. $\left(\frac{.701}{.813 \text{ mm}} \right)$

by bending the side electrode.

Valve Grinding and Cutting.

The operations detailed below are necessary for the purpose of ensuring a gas-tight seal between valve and seat in cylinder head.

Grinding-in is necessary whenever a new, refaced, or very slightly pitted valve is to be used in a cylinder head where the valve seat is in a similar condition. Valves and seats which have become more extensively worn should be replaced or re-cut before regrinding. The necessary cutting treatment depends entirely on the condition and previous treatment of the valve or seat, which should be examined and compared with the examples shown in Fig. 17.

Grinding-In.

It is most important that each valve is ground into its correct seat in the cylinder head, and for this reason it has been emphasised that care should be taken, after removal, to assemble valves in their correct order on the cylinder head Service Fixture. The valve grinding tool used can be either of the hand type or the pneumatic type as shown in Fig. 15, and the treatment for each valve is as follows:—

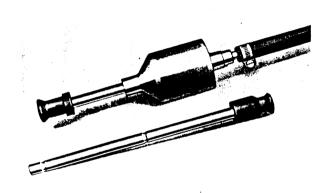


FIG. 15

- Mount the cylinder head on the the service stand with combustion chambers uppermost.
- 2. Coat the bevelled face of the valve with grinding paste. If the seats are in fairly good condition, it will only be necessary to use fine paste, but if this is not sufficient to produce a clean surface, a little coarse paste must be used, finishing off with fine grade.

- Exerting an even pressure on the grinding tool, rotate the valve backwards and forwards as shown in Fig. 16. After a few oscillations, lift the valve and press down in another position. This ensures even grinding.
- Examine the valve as the work proceeds until a smooth dull ring is formed round its seating face cor-



FIG. 16

responding with the seat cut in the cylinder head.

5. Test if seat is true. Carefully clean all grinding paste from both valve and seat, mark seat in at least 4 positions with a soft lead pencil, then rotate the dry valve as if continuing grinding process. The correctly ground-in valve will then make a bright ring concentric with the smooth matt band

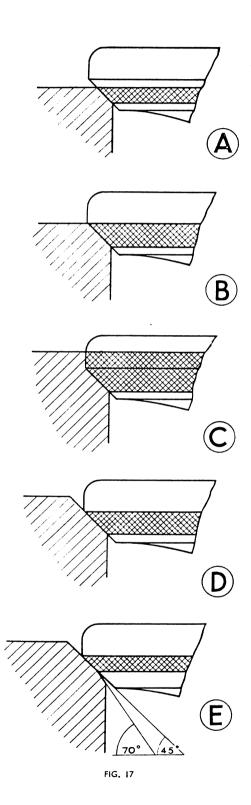
first observed. If this ring is unbroken and all pencil marks broken, the valve will be gas-tight on assembly.

Cutting Valves and Seats.

Typical conditions of valves and seats which require re-cutting are illustrated in Fig. 17.

Fig. 17A. shows a new, correctly seated valve providing a centrally located seating area of approximately half that of the inclined face. In Fig. 17B grinding has been carried out to such an extent that the valve head has sunk into the cylinder head, giving such a large seating area that its efficiency has been impaired.

Fig. 17C shows an extreme condition in which the valve is shrouded by a step formed on the seat due to excessive grinding. Engine performance will suffer because of the consequent later opening and early closing of the valve, and the loss of seating efficiency described above. In Fig. 17D the step illustrated in detail C has been removed, using the $44\frac{1}{2}$ ° cutter but the seating face still remains too large. Fig. 17E illustrates the use of the 70narrowing cutter to reduce the seating area to the correct size. It will be seen that the lower part of the $44\frac{1}{2}$ seat has been cut away, leaving the correct valve contact area of approximately half of the inclined face.



Refacing Valves.

Valve head seating angle is 90 included.

Valves are manufactured from a much harder steel than that of their seats in the cylinder head. Consequently, if a valve is badly burnt or pitted, the grinding operation previously described would remove an excessive amount of material from cylinder head seats before restoration of the sealing face of the valve heads. Therefore, badly pitted valves should always have their sealing faces re-ground at the correct angle before lapping-in. It is strongly recommended that grinding is carried out using the specialised machine illustrated in Fig. 18. The least possible amount of steel should be ground away, consistent with

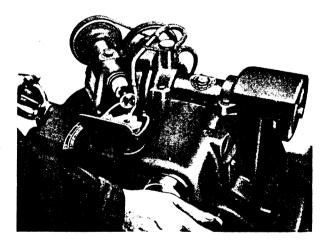


FIG. 18

the removal of the pittings. A valve should be discarded if re-facing treatment reduces its head thickness above the seating edge below $\frac{1}{32}$ " (I mm). If the head is too thin the edges are apt to curl up when the valve becomes hot.

Re-cutting Valve Seats.

Valve seat angle is 89° included.

It will not usually be necessary to recut seats with the $44\frac{1}{2}$ cutter unless they are badly pitted, or if a step has formed as shown in Fig. 17C use the cutter to remove the absolute minimum of metal necessary to form the profile shown in Fig. 17D. The narrowing cutter should be used as necessary to provide the correct sealing area shown in Fig. 17E.



FIG. 19

The use of the valve seat cutter and pilot, service tool FT.316, is illustrated in Fig. 19. The cutter is a push fit over the tapered shank of the tool, while the pilot is a sliding fit in the valve guide bore. After locating the pilot with the cutter teeth bearing lightly on the seat requiring treatment, a few revolutions of the tool will suffice to clean up the seating face. This procedure applies to either cutter.

Note—Valves and seats must be lapped-in after refacing. Valves should be mounted on the cylinder head service fixture in their correct order for assembly.

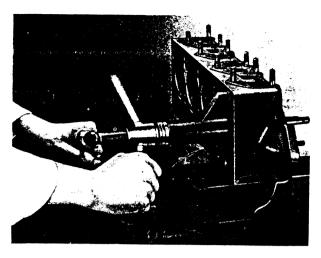


FIG. 20

Precautions before Re-Assembly.

Before re-assembling the eight ground-in valves, all traces of paste should be removed from valve head, stems, and seats in the cylinder head. Ensure that no paste remains in the valve guide bores by dipping valve stems in petrol and passing up and down in guides. Ensure that all valve guides protrude $\frac{9}{16}$ " (14.3 mm) above valve spring seats in cylinder head. If the protrusion is less than $\frac{9}{16}$ " (14.3 mm) the guide will have to be removed, cleaned, and re-located, using service tool FT.60—Fig. 20 and 21. Examine all valve springs. If a spring compresses to a length less than $1\frac{1}{4}$ " (31.75 mm) under a load of 38 lbs. (17.237 kg.) it should be renewed.

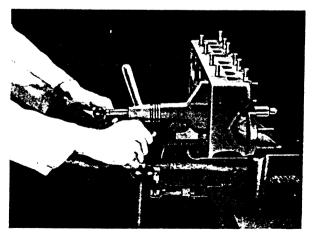


FIG. 21

Assembly of Valves and Springs.

Locate each valve in its correct guide and mount the cylinder head on the service stand with valve spring seats uppermost. Oil retaining cups and seals are not to be fitted.

Assemble each valve spring as follows:-

- 1. Place spring over the valve stem with close wound coils to spring seat shown in Fig. 23 inset.
- 2. Mount collar and compress spring.
- 3. Insert split cones, and allow the spring to expand.

To replace Cylinder Head Assembly.

- I. Remove all rag from cylinder bores.
- 2. Ensure that none of the tappets has become displaced in its bore.

- 3. Remove cylinder sleeve retainers.
- 4. Carefully place a new cylinder head gasket over studs on to face of block.

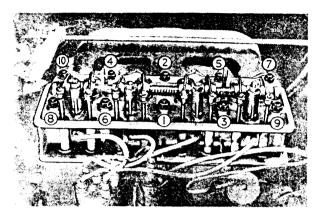
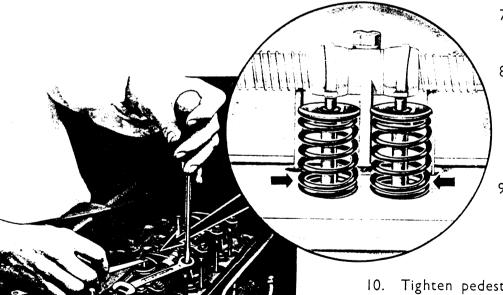


FIG. 27

- 5. Mount cylinder head over studs and tighten nuts over washers gradually in order shown in Fig. 22 to a torque wrench reading of 60—65 ft. Tbs. (8.25—8.95 m.kg.)
- 6. Insert push rods in their tubes.

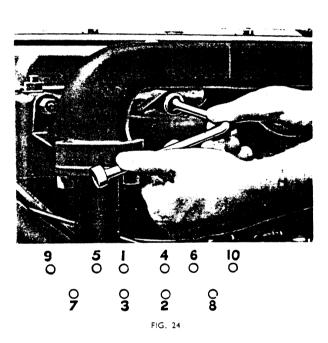


- 7. Slacken offall rocker adjusters.
- Mount rocker shaft assembly, locating pedestals over the four long studs on left side of head.
- Locate rocker ends over valve tips, and adjusters in push rod cups.
- 10. Tighten pedestal securing nuts.
- Adjust all valve tip clearances to 0.010" (.25 mm) inlet, 0.012 (.3 mm) exhaust by means of rocker adjuster screws and lock nuts as shown in Fig. 23.

- 12. Replace rocker cover with gasket.
- 13. Replace spark plugs and leads.

To Complete Re-Assembly.

- Renew manifold gasket if showing signs of "blowing."
- 2. Mount manifold on studs over gaskets, and secure by tightening nuts on to lock washers. Tighten gradually in the order shown in Fig. 24.



- Attach exhaust pipe clip to engine mounting flange by tightening set screws.
- 4. Secure thermostat body to flange at forward end of cylinder head by tightening two set screws on to lock washers.
- 5. Re-connect thermostat by-pass hose at thermostat body and radiator hose and stay at water outlet elbow.
- 6. Replace governor spring link and secure with cotter pin.

- 7. Replace crankcase breather pipe with fibre washers at screwed adaptor in manifold and banjo connection on rocker cover.
- 8. Replace fuel tank and re-connect fuel pipe at sediment bowl.
- 9. Replace hood.
- 10. Replace battery.
- 11. Re-fill radiator.
- 12. Turn on petrol.

After the engine has been run for a few hours, re-adjustment of valve tip clearances may be necessary, due to bedding-in of valves.

CAMSHAFT AND TIMING MECHANISM

Removal of Timing Cover.

- 1. Support engine forward of sump plug and remove hood together with front axle and radiator assembly.
- 2. Remove fan belt and fan which is secured to pulley by 4 set screws with lock washers.

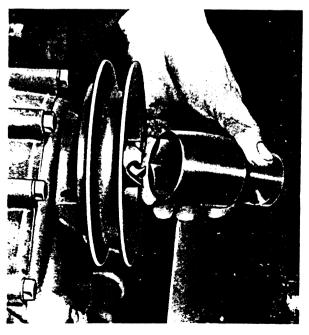


FIG. 25

- 3. Remove starting handle jaw, using service tool FTB.16, Fig. 25, with long lever. Bend back tabs of starting handle jaw lock washer, then select 4th gear and apply parking brake before striking lever to unscrew jaw.
- 4. Remove pulley from crankshaft key.
- 5. Disconnect throttle tie rod from governor lever by removal of clevis:
- 6. Disconnect governor lever rod from ball connection on lever at forward end of throttle rod.
- 7. Pull timing cover with gasket free from the two locating dowels, after removal of 3 bolts, 8 set screws, and one nut from stud.

Removal of Camshaft.

- I. Withdraw governor cup and shaft assembly from its hole in the camshaft Then remove governor plate for examination. See Section F.
- 2. Bend back corners of locking plate and remove the two set screws securing camshaft timing sprocket.
- 3. Remove crankshaft sprocket oil thrower, and lever crankshaft sprocket about $\frac{1}{8}$ " forward.
- 4. Remove camshaft sprocket with chain.
- 5. Remove camshaft locating plate which is secured to engine plate by 3 set screws on lock washers.
- 6. Remove rocker assembly, cylinder head, pushrod and tappets. (See page C7—11).
- 7. Withdraw camshaft.

Re-assembly—Valve Timing.

- Replace camshaft, tappets, cylinder head, push rods and rocker mechanism. (See page C17), for cylinder head replacement carry out instruction 1—10 only.
- 2. Locate crankshaft sprocket.
- 3. Replace camshaft sprocket. (See "Valve Timing").
- 4. Before adjusting valve timing, check sprocket alignment by placing a straight-edge across front faces of sprockets and shim as necessary behind crankshaft sprocket. This check should always be made if it is suspected that the engine has had treatment which has affected crankshaft or camshaft end float.
- Replace crankshaft sprocket oil thrower with dished edge forward and timing cover with new gasket.
- 6. Replace pulley and fan, locating any balancer weights with the stamped "BALANCER" to front and with drilled holes in line. Note that fan blades should be to rear of central mounting plate. Should stud have screwed out when removing timing cover, it should be replaced directly below water pump.

Valve Timing.

The following procedure assumes that adjacent teeth of camshaft and crankshaft timing sprockets have been scribe marked when No. I inlet and exhaust valves are fully closed with No. 4 inlet valve just opening and exhaust valve just closing.

- Set No. I piston at T.D.C. See Section G Pages 17-19.
- 2. Locate camshaft sprocket by set screws through any two convenient holes.
- 3. Turn sprocket and camshaft until scribe marks are adjacent.
- 4. Lightly press No. 4 cylinder rocker adjusting screw into push rod cups, and rock camshaft sprocket gently backwards and forwards as shown in Fig. 26 to check whether the introductory condition applies.

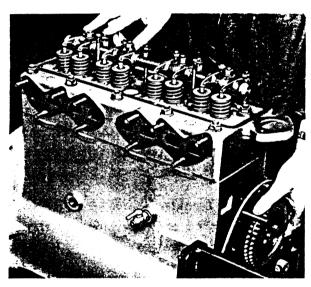


FIG. 26

- 5. If instruction (4) gives a positive result, mount chain and secure camshaft sprocket in position chosen.
- 6. If instruction (4) gives a negative result, repeat instructions (2) to (5) having located camshaft sprocket 90° clockwise from previous position. There are four alternative positions.
- 7. Adjust all rocker clearances to—
 0.010" (.254 mm) inlet.
 0.012" (.305 mm) exhaust.
 replace cover with new gasket before continuing re-assembly.

The following procedure assumes that no components are marked, in which case it is necessary to remove starter for access to flywheel. The operations can, of course, be more conveniently carried out if engine is removed and flywheel completely exposed.

- 1. Mount and locate camshaft sprocket and chain by inserting set screws through any two convenient holes.
- 2. Turn flywheel by means of a suitable lever through starter hole in crankshaft flange, until No. 4 inlet and exhaust valves are closed, then adjust their rocker clearances to:

0.020" (.508 mm) inlet. 0.022" (.559 mm) exhaust.

- 3. Insert a 0.010" (.254 mm) feeler gauge between rocker and No. 4 inlet valve tip and turn flywheel in direction of normal rotation, until feeler is just beginning to be gripped (i.e. valve beginning to open).
- 4. Chalk mark flywheel and adjacent spot on crankcase.
- 5. Insert 0.010" (.254 mm) feeler gauge between rocker and No. 4 exhaust valve tip and turn flywheel in same direction until grip on feeler is just beginning to relax (i.e. valve just closing).
- 6. Chalk mark flywheel adjacent to previous mark on crankcase.
- 7. Return flywheel to original position and, approaching in direction of normal rotation, position it so that its two chalk marks are equidistant from the one on the crankcase. This positions the camshaft so that No. 4 inlet valve

has just begun to open and No. 4 exhaust valve is just about to close, with No. I inlet and exhaust valves fully closed.

 Without altering position of camshaft, remove sprocket and chain and turn flywheel until No. I piston is at T.D.C. (See Section G. Pages 17-19)

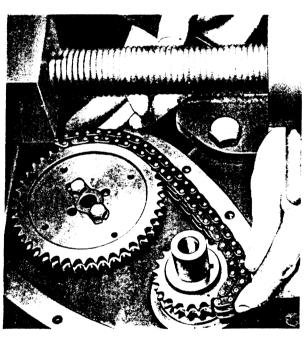


FIG. 27

- 9. Still without altering position of camshaft, mount sprocket with set screws finger tight and apply chain as shown in Fig. 27. The chain, located on crankshaft sprocket, should engage at least 4 teeth on camshaft sprocket with no slackness. Reversal of sprocket provides four alternative positions on camshaft, giving a location variable by $\frac{1}{4}$ tooth. The drillings in camshaft sprocket are located so that 90° movement will give $\frac{1}{2}$ tooth adjustment, while reversal of sprocket will give a further $\frac{1}{4}$ tooth adjustment in either direction.
- 10. Having found the correct position for camshaft sprocket, mount chain, adjust all rocker clearances to:—

0.010" (.254 mm) inlet.

0.012" (.305 mm) exhaust.

replace cover with new gasket before continuing re-assembly.

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